

Lecture 10: Multiple degrees of freedom system using modal superposition (Optional)

Reading materials: Section 3.8

1. Equations of motion

$$\mathbf{m} \ddot{\mathbf{u}} + \mathbf{c} \dot{\mathbf{u}} + \mathbf{k} \mathbf{u} = \mathbf{f}(t); \quad \mathbf{u}(0) = \mathbf{u}_0; \quad \dot{\mathbf{u}}(0) = \mathbf{v}_0$$

$$\mathbf{f}(t) = \mathbf{F} \sin(\omega_f t)$$

 Rayleigh damping

$$\mathbf{c} = \alpha \mathbf{m} + \beta \mathbf{k}$$

2. Uncoupled equations

 Undamped free vibration mode shapes and frequencies

$$\mathbf{k} \boldsymbol{\phi}_i = \lambda_i \mathbf{m} \boldsymbol{\phi}_i; \quad i = 1, 2, \dots, n$$

$$\boldsymbol{\phi}_j^T \mathbf{m} \boldsymbol{\phi}_i = 0; \quad i \neq j \quad \boldsymbol{\phi}_j^T \mathbf{k} \boldsymbol{\phi}_i = 0; \quad i \neq j$$

 Uncoupled equations

$$\mathbf{z} = (z_1 \ z_2 \ \dots \ z_n)^T$$

$$\mathbf{u}(t) = \sum_i z_i(t) \boldsymbol{\phi}_i$$

$$M_i \ddot{z}_i(t) + (\alpha M_i + \beta K_i) \dot{z}_i(t) + K_i z_i(t) = R_i \sin(\omega_f t); \quad i = 1, 2, \dots$$

$$z_i(0) = \frac{1}{M_i} (\phi_i^T \mathbf{m} u^0); \quad \dot{z}_i(0) = \frac{1}{M_i} (\phi_i^T \mathbf{m} v^0);$$

$$M_i = \phi_i^T \mathbf{m} \phi_i; \quad K_i = \phi_i^T \mathbf{k} \phi_i; \quad \omega_i = \sqrt{K_i/M_i}; \quad R_i = \phi_i^T \mathbf{F}$$

$$C_i = \alpha M_i + \beta K_i \equiv 2 M_i \xi_i \omega_i$$

$$M_i \ddot{z}_i(t) + C_i \dot{z}_i(t) + K_i z_i(t) = R_i \sin(\omega_f t); \quad i = 1, 2, \dots$$

• Solution

$$\mathbf{u}(t) = \sum_i z_i(t) \phi_i$$

$$z_i(t) = e^{-\xi_i \omega_i t} (A_i \cos \omega_{di} t + B_i \sin \omega_{di} t) + \frac{R_i/K_i}{\gamma_i} [(1 - r_i^2) \sin \omega_f t - 2 \xi_i r_i \cos \omega_f t]$$

$$r_i = \frac{\omega_f}{\omega_i}$$

$$\gamma_i = (1 - r_i^2)^2 + (2 \xi_i r_i)^2$$

$$A_i = u_0 + \frac{2 R_i r_i \xi_i}{K_i \gamma_i};$$

$$B_i = \frac{\xi_i \omega_i u_0}{\omega_{di}} + \frac{v_0}{\omega_{di}} - \frac{R_i ((1 - r_i^2) \omega_f - 2 r_i \omega_i \xi_i)}{K_i \gamma_i \omega_{di}}$$