

THE UNIVERSITY OF IOWA  
Department of Mechanical Engineering

Fracture Mechanics  
ME:5159

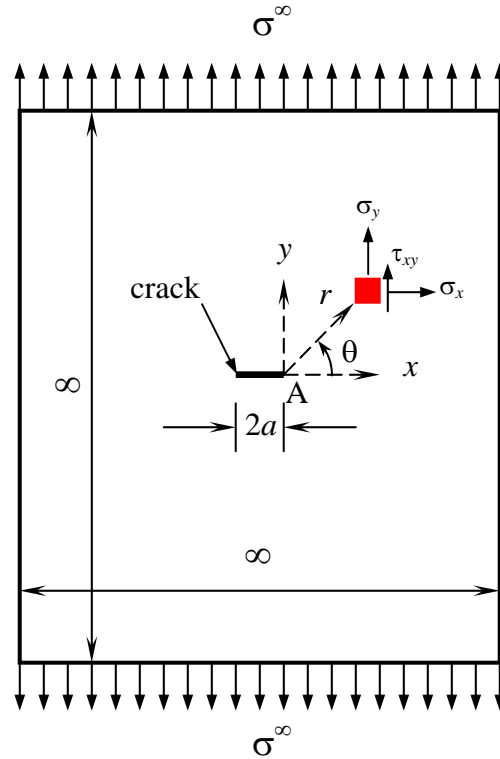
Homework #4  
Total Points: 30

Assigned: March 25, 2020  
Due: April 03, 2020

**Problem 1:**

A weightless, homogeneous, isotropic, infinite plate with a crack of length  $2a$  is subjected to a far-field applied stress  $\sigma^\infty$ , as shown in the figure below. With the crack-tip A as the origin of the coordinate system, the linear-elastic crack-tip stress field (*i.e.*, when  $r \rightarrow 0$ ) can be obtained as

$$\begin{aligned}\sigma_x &= \frac{K_I}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left[ 1 - \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right] \\ \sigma_y &= \frac{K_I}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left[ 1 + \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right], \quad 0 \leq |\theta| \leq \pi \\ \tau_{xy} &= \frac{K_I}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \sin \frac{\theta}{2} \cos \frac{3\theta}{2}\end{aligned}$$



where  $K_I = \sigma^\infty \sqrt{\pi a}$  is the mode-I stress intensity factor. For the plate material with uniaxial yield strength  $S_y$ , let  $r_y(\theta)$  denote the plastic zone size as a function of  $\theta$ . Using the linear-elastic stress field above and assuming no redistribution of stresses due to crack-tip plasticity,

- Determine the plastic zone size  $r_y(\theta)$  in terms of  $K_I$  and  $S_y$  for (1) plane stress and (2) plane strain conditions.
- From the results of (a) and  $\nu = 1/3$ , sketch the plastic zone boundary in terms of  $r_y(\theta) \cos \theta / [K_I^2 / (\pi S_y^2)]$  vs.  $r_y(\theta) \sin \theta / [K_I^2 / (\pi S_y^2)]$  plot at crack tip A for  $0 \leq |\theta| \leq \pi$ . Which stress state gives conservative prediction of the plastic zone size? Comments.

Assume *von Mises yield criterion* for your analysis.

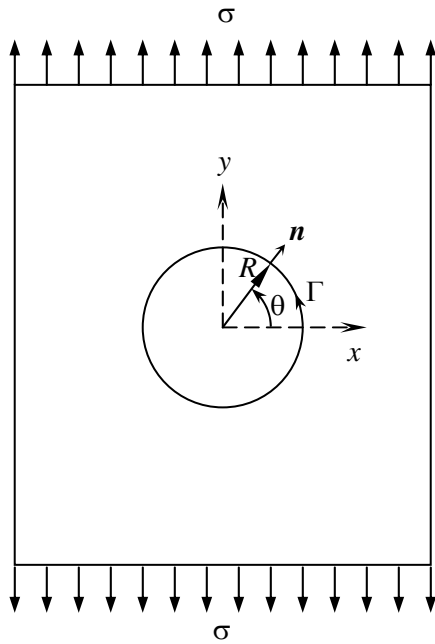
**Problem 2:**

Solve Problem 1 assuming *Tresca yield criterion*. Compare results of Problems 1 and 2. Comments.

**Problem 3:**

Consider an uncracked, linear-elastic ( $E, \nu$ ) specimen in plane stress, which is subject to uniaxial tensile stress of magnitude  $\sigma$ , as shown in the figure below. Let  $\Gamma$  denote a closed circular contour of radius  $R$  indicated in the figure. Confirm that the  $J$ -integral for this uncracked body is

$$J = \int_{\Gamma} \left[ W dy - \mathbf{T} \cdot \frac{\partial \mathbf{u}}{\partial x} d\Gamma \right] = 0.$$



$$\mathbf{T} = \boldsymbol{\sigma} \mathbf{n}$$

$$\boldsymbol{\sigma} = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{zx} \\ \tau_{xy} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{yz} & \sigma_z \end{bmatrix}; \quad \mathbf{n} = \begin{Bmatrix} n_x \\ n_y \\ n_z \end{Bmatrix}$$

$$\mathbf{u} = \begin{Bmatrix} u_x \\ u_y \\ u_z \end{Bmatrix}$$