Problem 1: Shear stress (Chapter 1) Information and assumptions

- $v = 1.28 \times 10^{-2} ft^2/s$
- $\rho_{water} = 1.94 \ slugs/ft^3$

- $u(y) = \frac{B}{2\mu}(y^2 hy) + V\left(1 \frac{y}{h}\right)$
- V = 0.02 ft/s
- h = 1.0 in
- $B = -0.334 \, lb/ft^3$
- $A = 100 ft^2$



Find

• Find (a) shear stress on the plate, (b) required force, and (c) power to pull the plate

Solution

(a) Calculate dynamic viscosity

$$\mu = \nu \cdot \rho = \nu \cdot (SG \cdot \rho_{water}) = (1.28 \times 10^{-2})(1.26 \times 1.94) = 3.13 \times 10^{-2} \text{ lb} \cdot \text{s/ft}^2 + 1.23 \times 10^{-2} \text{ lb} \cdot \text{s/ft}^2$$

Shear stress

$$\tau = \mu \frac{du}{dy}$$
$$\tau = \mu \left(\frac{B}{2\mu}(2y - h) - \frac{V}{h}\right)$$

Shear stress at the wall (at y=0)

$$\therefore \tau = (3.13 \times 10^{-2}) \left(\frac{(-0.334)}{(2)(3.13 \times 10^{-2})} \left(2 \times 0 - \frac{1}{12} \right) - \frac{(0.02)}{(1/12)} \right) = \mathbf{0}.\,\mathbf{0064}\,\,\mathbf{lb/ft^2} + \mathbf{7}$$

(b) Friction force

$$F = \tau \cdot A = (0.0064)(100) = 0.64 \text{ lb} + 1$$

(c) Power

$$P = F \cdot V = (0.64)(0.02) = 0.0128 \text{ lb} \cdot \text{ft/s} + 1$$

Problem 2: Hydrostatic force (Chapter 2) Information and assumptions

- Length of barrier 8 m
- $\gamma = 9.80 \ kN/m^3$

Find

• Find (a) the horizontal force (magnitude F_H and location y_cp) and vertical force (magnitude F_V) exerted on the curved surface of the barrier and (c)the barrier weight W

Solution

(a) Horizontal pressure force

$$F_H = \gamma h_c A$$

$$F_H = (9.80)(0.5)(8 \times 1) = 39.2 \text{ kN} + 3$$

Location of the horizontal force

$$y_{cp} = \bar{y} + \frac{I}{\bar{y}A}$$

$$y_{cp} = 0.5 + \frac{(8)(1)^3/12}{(0.5)(8 \times 1)} = 0.5 + 0.167 = 0.667 \text{ m} + 3$$

(b) Vertical pressure force

$$F_V = \gamma \Psi$$

 $F_V = (9.80)(\pi)(0.5)^2(8)\left(\frac{1}{2}\right) = 30.8 \text{ kN} + 3$

(c) Since the contact is frictionless the barrier weight must balance the vertical hydrostatic force,

$$\sum F = F_V - W = 0$$
$$\therefore W = 30.8 \text{ kN} + 1$$



Problem 3: Bernoulli equation (Chapter 3)

Information and assumptions

• $\gamma = 62.4 \ lb/ft^3$

- Ignore friction loss
- SG = 13.6 for the manometer fluid
- $g = 32.2 ft/s^2$

Find

• Determine (a) the volume flow rate Q and (b) the waterjet velocity V_i leaving nozzle



Solution

(a) Bernoulli equation

$$p_1 + \frac{1}{2}\rho V_1^2 + z_1 = p_2 + \frac{1}{2}\rho V_2^2 + z_2$$

Applying assumptions $z_1 = z_2$ and $V_2 = 0$

$$p_1 + \frac{1}{2}\rho V_1^2 = p_2 + 5$$

Also, for the manometer,

$$p_2 - p_1 = \gamma(SG - 1)(1 \text{ in.}) = (62.4)(13.56 - 1)\left(\frac{1}{12}\right) = 65.3 \text{ lb/ft}^2 + 2$$

By solving the Bernoulli equation for V_1 ,

$$V_1 = \sqrt{2\left(\frac{p_2 - p_1}{\rho}\right)} = \sqrt{(2)\frac{(65.3)}{62.4/32.2}} = 8.21 \text{ ft/s}$$

Thus,

$$Q = V_1 A_1 = (8.21) \left(\frac{\pi}{4}\right) (1)^2 = 6.45 \text{ ft}^3/\text{s} + 2$$

(b) From continuity

$$V_j = \frac{Q}{A_j} = \frac{6.45}{\left(\frac{\pi}{4}\right) \left(\frac{3}{12}\right)^2} = 131.4 \text{ ft/s} + 1$$

Problem 4: Acceleration and Euler equation (Chapter 4) Information and assumptions

- Two dimensional flow
- $\underline{V} = U(1 a^2/x^2)\hat{\imath}$
- $\overline{\rho}a_x = -dp/dx$
- $\rho = 917 \, kg/m^3$
- U = 2 m/s
- a = 6 cm

Find

Calculate (a) the fluid velocity u, (b) the acceleration a_x, and (c) the pressure gradient dp/dx at point 1



(a) Velocity

$$u = U\left(1 - \frac{a^2}{x^2}\right)_{x = -2a} = (2)\left(1 - \frac{a^2}{(2a)^2}\right) = (2)\left(1 - \frac{1}{4}\right) = 1.5 \text{ m/s} + 2$$

(b) Acceleration

$$a_{x} = \frac{\partial u}{\partial t} + u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} + w\frac{\partial u}{\partial z}$$

Since u is a function of x only

$$a_x = u \frac{\partial u}{\partial x}$$
$$a_x = U \left(1 - \frac{a^2}{x^2} \right) U \left(\frac{2a^2}{x^3} \right) = \frac{2U^2}{a} \left(\frac{a^3}{x^3} - \frac{a^5}{x^5} \right)$$

At x = -2a,

$$\therefore a_{x} = \frac{2U^{2}}{a} \left(\frac{a^{3}}{(-2a)^{3}} - \frac{a^{5}}{(-2a)^{5}} \right) = -\frac{3U^{2}}{16a} = -\frac{(3)(2)^{2}}{(16)(0.06)} = -12.5 \text{ m/s}^{2} + 6$$

(c) Pressure gradient

$$\frac{dp}{dx} = -\rho a_x = -(917)(-12.5) = 11462.5 \text{ N/m}^3 = 11.5 \text{ kPa/m}$$

