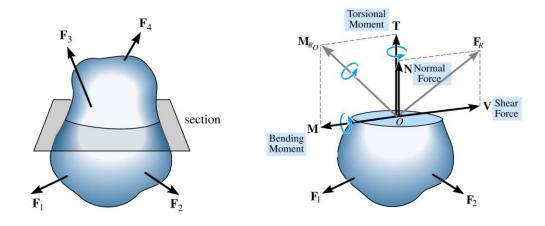
# **Period #2 : Average Normal Stresses**

#### A. Review of Statics

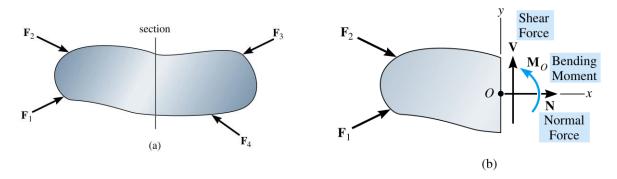
When external loads are applied to a restrained deformable body, internal forces are generated to keep the body in equilibrium.

The internal forces in a body can be examined using methods of sectioning and principles of statics.



On a given section, the normal force **N** and the bending moment **M** give rise to normal stresses.

#### **System of Co-Planar Forces**



Summary: Using methods of statics we can determine the internal <u>forces</u> and <u>moments</u>.

#### **B. Stress**

On a section through a body, the internal forces and moments are created by stresses.

At a point on a section, stress is force per unit area.

Forces per unit area that act normally to the section are called <u>normal stresses</u>. In this course, normal stresses are denoted by the symbol  $\sigma$ .

Forces per unit area that act tangentially to the section are called <u>shear stresses</u>. Shear stresses in this course are denoted by the symbol  $\tau$ .

Consider the body shown below with a cut section having a unit normal in the z-direction:

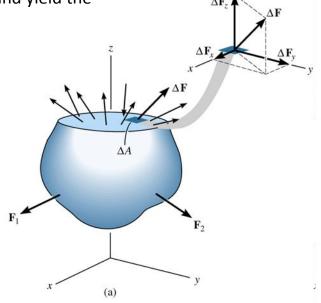
The force acting force on an infinitesimal area  $\Delta A$ , is  $\Delta F$ .

The z-component of force  $\Delta {\bf F}$  acts normally to the section and yields the normal stress component  $\sigma_{_{7}}$ 

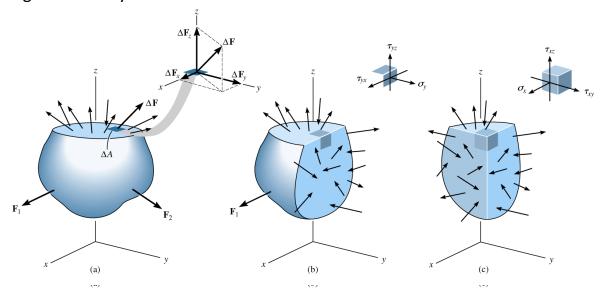
$$\sigma_z = \frac{\Delta F_z}{\Delta A}$$

The x- and y-components of force  $\Delta F$  act tangentially to the section and yield the shear stress components  $\tau_{zx}$  and  $\tau_{zy}$  respectively.

$$\tau_{zx} = \frac{\Delta F_x}{\Delta A}$$
 $\tau_{zy} = \frac{\Delta F_y}{\Delta A}$ 



More generally, the figure below shows the stress components at the same point when sections orthogonal to the y- and x-axes are also considered.



For an infinitesimal area  $\Delta A$  orthogonal to the y-axis with force components  $\Delta F_y$ ,  $\Delta F_x$ , and  $\Delta F_z$  acting upon it:

$$\sigma_{y} = \frac{\Delta F_{y}}{\Delta A}$$
  $\tau_{yx} = \frac{\Delta F_{x}}{\Delta A}$   $\tau_{yz} = \frac{\Delta F_{z}}{\Delta A}$ 

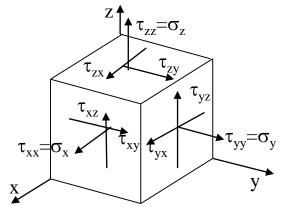
For an infinitesimal area  $\Delta A$  orthogonal to the x-axis with force components  $\Delta F_x$ ,  $\Delta F_y$ , and  $\Delta F_z$  acting upon it:

$$\sigma_{x} = \frac{\Delta F_{x}}{\Delta A}$$
  $\tau_{xy} = \frac{\Delta F_{y}}{\Delta A}$   $\tau_{xz} = \frac{\Delta F_{z}}{\Delta A}$ 

The common definition of stress is <u>force</u> per unit <u>area</u>.

To be precise, we need to be very specific about the <u>directionality</u> of the forces and the orientation of the area on which the forces are acting. This is done by representing stress as a rank-2 tensor as follows:

$$\mathbf{\tau} = \begin{bmatrix} \tau_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \tau_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \tau_{zz} \end{bmatrix} = \begin{bmatrix} \sigma_{x} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{y} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{z} \end{bmatrix}$$



The diagonal entries in the stress tensor are said to be **normal stresses** because the associated force component acts normal to the plane.

The off-diagonal entries in the stress tensor are called **shear stresses**, because the associated force components act parallel to the plane

#### A few points about stress:

The dimensions of stress are FL<sup>-2</sup> where F denotes a unit of force and L denotes a unit of length. Examples are:

N/m<sup>2</sup> where N is a newton and m denotes a meter.

One newton per square meter is called 1 pascal or 1 Pa. lb/in<sup>2</sup> or pounds per square inch (psi).

Normal stress, in this course, is taken positive in tension and negative in compression.

Stresses are point-wise quantities that generally vary over a section in a body.

Sometimes, we are interested in the <u>average</u> stresses acting on a given section. The average normal stress on a given section is defined as follows:

$$\overline{\sigma} = \frac{\int \sigma \ dA}{A}$$

Though we should, we will not always use the overbar to represent average normal stresses.

### C. Average Normal Stress in Axially Loaded Members

Axial members by definition possess a longitudinal axis.

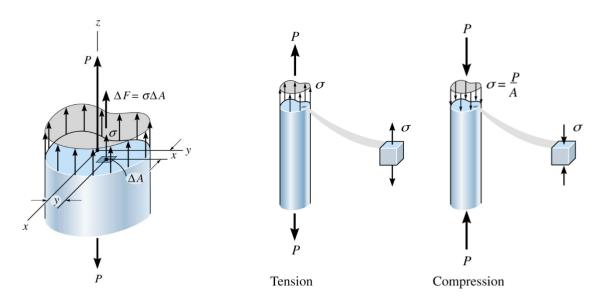
Cross-sections of axial members have normals that are parallel to the longitudinal axis.

Purely axially loaded members undergo strictly axial loading. The resultant normal force at any cross-section passes through the centroid of that section.

•The average normal stress at a cross-section of an axially loaded member is:

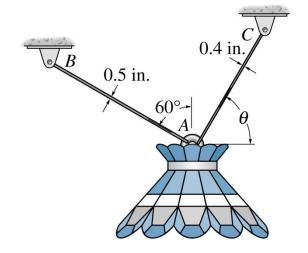
$$\sigma = \frac{P}{A}$$

where P is the axial force at the given cross-section, and A is the area of the given cross-section.

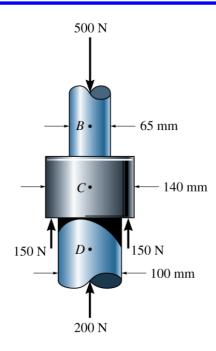


## **D.** Example Problems

**Example 2.1** The 50-lb lamp is supported by the two-rods shown. Determine the angle of orientation q of member AC such that the average normal stress in rod AC is twice the average normal stress in rod AB. What is the magnitude of stress in each rod?



**Example 2.2** The thrust bearing is subjected to the loads shown. Determine the average normal stress developed on cross sections through points B, C, and D.



**Example 2.3** The joint shown is subjected to the axial member force of 6 kip. Determine the average normal stress acting on sections AB and BC. Assume the member is *smooth* and is 2 inches thick.

