Chapter 6: Turbulent Transport and its Modeling

Part 1: Molecular Momentum Transport

 $\rho \overline{u_i u_i} =$ turbulent momentum flux

 $\rho \overline{uv} = x$ momentum ρu in y direction due to turbulent v, for V = (U + u, v, w)

Classical ideas for modeling turbulent transport were based on molecular momentum transport for ideal (non-dense) gas: molecules far apart and intermolecular forces are weak. Molecules in free flight with brief collisions at which time their direction and speed change.



Figure 6.1 Molecular model of the viscosity of a gas.

Across plane separating the gas in two regions, the molecules do not attract or repel each other (contrasting to liquids); therefore, the primary source of shear stress is that due to microscopic transport of momentum due to random molecular motions.

Newtonian fluid stress rate of strain relationship

$$\begin{aligned} \pi_{21} &= \mu \left(\frac{\partial v_1}{\partial x_2} + \frac{\partial v_2}{\partial x_1} \right) \\ &= \mu \frac{\partial v_1}{\partial x_2} \end{aligned} \quad \text{For } \underline{V} = v_1(x_2)\hat{e}_1 \end{aligned}$$

Which can be derived for ideal gas using four facts from kinetic theory:

- 1) Molecules that cross the plane $x_2 = \text{constant begin their free flight on}$ average at distance $\pm \frac{2}{3}l$ (l = mean free path) from the plane.
- 2) Mean free path f(d, n):

 $l = 1/\sqrt{2}\pi d^2 n$

d = molecular diameter n = number density of molecules per unit volume

Since molecules have a distribution of speeds and they are moving relative to each other, $RHS = \frac{3}{4}RHS$ or if assume Maxwellian distribution of velocities RHS = 0.707·RHS.

- 3) Flux of velocities across x_2 -plane per unit area $=\frac{1}{4}n\overline{v}$, where \overline{v} = average molecular speed (without regard direction).
- 4) Average molecular speed

$$\overline{v} = \left[\frac{8kT}{\pi m}\right]^{1/2} = f(T)$$

$$k = \text{Boltzmann constant}$$

$$m = \text{molecular mass}$$

Shear stress x_2 -plane

$$\tau_{21} = \frac{x_1 \text{ force}}{\text{unit area}} = \text{net flux momentum across } x_2 \text{ plane}$$

x-momentum of one particle from above:

$$mv_1|_{x_2+\frac{2}{3}l} = m\left[v_1 + \frac{dv_1}{dx_2}\left(\frac{2}{3}l\right) + \cdots\right]_{x_2}$$

x-momentum of one particle from below:

$$mv_1|_{x_2-\frac{2}{3}l} = m\left[v_1 + \frac{dv_1}{dx_2}\left(-\frac{2}{3}l\right) + \cdots\right]_{x_2}$$

Net flux = difference between the momentum of a particle from above minus one particle from below times the rate that the particles cross a unit area $\frac{1}{a}n\overline{v}$:

$$\tau_{21} = \frac{1}{4} n\overline{v} \times \left[mv_1 \big|_{x_2 + \frac{2}{3}l} - mv_1 \big|_{x_2 - \frac{2}{3}l} \right]$$
$$= \frac{1}{4} n\overline{v} \times m \frac{4}{3} l \frac{dv_1}{dx_2}$$

i.e., comparing with $\tau_{ij} = \mu(v_{i,j} + v_{j,i})$

$$\mu = \frac{1}{3}n\overline{\overline{\nu}}ml = \frac{2}{3d^2} \left[\frac{mkT}{\pi^3}\right]^{1/2} = \frac{1}{3}\rho\overline{\nu}l$$

= $f(\text{molecular properties and T})=f(\overline{v}, l)$

 $\mu \uparrow m \uparrow$

- $\mu \downarrow d \uparrow$
- $\mu \neq f(p)$

 $\mu \uparrow \sqrt{T} \uparrow$

More complete theory includes inter molecular forces and better agreement T dependence.

Viscous liquids need more advanced models considering intermolecular forces, but results in same $\tau_{ij} = \mu(v_{i,j} + v_{j,i})$ relationship.

Modelling Turbulent Transport by Analogy to Molecular Transport

Newtonian fluids (incompressible flow)

$$\sigma_{ij} = -p\delta_{ij} + 2\mu\varepsilon_{ij}$$
$$= -p\delta_{ij} + \tau_{ij}$$

$$\varepsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$$

$$\mu = \text{isotropic viscosity}$$

$$= \text{property of the fluid}$$

μ

In analogy the turbulent Reynolds stresses are modeled using the eddy viscosity concept

$$a_{ij} = -\rho \overline{u_i u_j} + \frac{2}{3}\rho k \delta_{ij} = \rho v_t \underbrace{\left(\overline{U_{i,j}} + \overline{U_{j,i}}\right)}_{\text{Mean flow rate of strain}} = \rho v_t S_{ij}$$

Anisotropic RS is modeled using isotropic eddy viscosity v_t or $\mu_t = \rho v_t$, which may be contrasted with μ definition for ideal gas; however, no reason to believe turbulent motions are without directional biases that are not aligned with S_{ii} .

Nonetheless eddy viscosity concept forms the basis of traditional RANS modeling, which focuses on modeling of v_t .

For example, consider 1D shear flow:

$$\frac{\tau_{12}}{\rho} = -\overline{uv} = v_t \frac{d\overline{U}}{dy}$$

Large scale turbulent eddies are most important in transporting momentum across the flow, which are mostly driven by inertia and pressure forces vs. viscosity. Assume $-\rho \overline{uv}$ due to turbulent eddies with transverse size *l* and intensity characterized by velocity scale u_0 .



Figure 26.4 Mean velocity profile and the fluctuations that contribute to the Reynolds stresses.

Therefore, $\overline{uv} < 0$, as per previous discussion Chapter 3 RANS equations.

Assume $A_p \rightarrow B$ and viceversa $B_p \rightarrow A$, where p = fluid particles, which interact and merge with the flow (transporting momentum)

$$-\rho \overline{uv} = f\left(\rho, l, u_0, \frac{du}{dy}\right) \\ -\frac{\overline{uv}}{u_0^2} = f\left(\frac{l}{u_0}\frac{du}{dy}\right)$$

Dimensional analysis

Assume linear relationship and eddy viscosity:

$$-\overline{uv} = Cu_0^2 \frac{l}{u_0} \frac{du}{dy}$$
$$= Clu_0 \frac{du}{dy}$$

i.e.,

$$v_t = C l u_0$$

 $u_0 =$ turbulent velocity scale l = turbulent length scale Which is also consistent with ideal gas theory:

$$\mu = \frac{1}{3}\rho \overline{u}l$$

The time scale for eddy turnover times is:

 l/u_0

And time scale for mean flow

$$\left|\frac{du}{dy}\right|^{-1}$$

Therefore assume:

$$\frac{l}{u_0} = \left|\frac{du}{dy}\right|^{-1}$$

i.e.,

$$u_0 = l \left| \frac{du}{dy} \right|$$

$$v_t = C l u_0 = C l^2 \left| \frac{du}{dy} \right|$$

Or

$$\mu_t = C_\mu \rho \ l^2 \left| \frac{du}{dy} \right|$$

1) This approach is the Prandtl mixing length theory. l depends on the type of flow.

 $l \propto \text{larger scale eddies}$

Free shear-flow: $l = c\delta$ where c = f (mixing layer, jet, wake),

BL: l = ky, i.e., eddy size $\propto y$ near wall

 $= c\delta$ away from the wall

 δ = appropriate width of viscous flow

2) $k - \varepsilon$ model

$$u = \sqrt{k}$$
 and $l = k^{3/2}/\varepsilon$ = length associated eddy turnover time l/u

$$\mu_t = C_\mu \rho \frac{k^2}{\varepsilon} \rightarrow \text{additional equations needed to model } k \text{ and } \varepsilon.$$

Eddy viscosity concept is based on ideal gas molecular transport; thus, assumes:

- 1) Mixing occurs over well-defined mixing time.
- 2) Momentum preserved between collisions.
- 3) Linear velocity variation over the mixing length.