

The exam is closed book and closed notes.

When the pump in the Figure below draws $220 \text{ m}^3/\text{hr}$ of water at 20°C ($\rho = 998 \text{ kg/m}^3$) from the reservoir, the total friction head loss is 5 m . The flow discharges through a nozzle to the atmosphere. Estimate the pump power in kW delivered to the water.

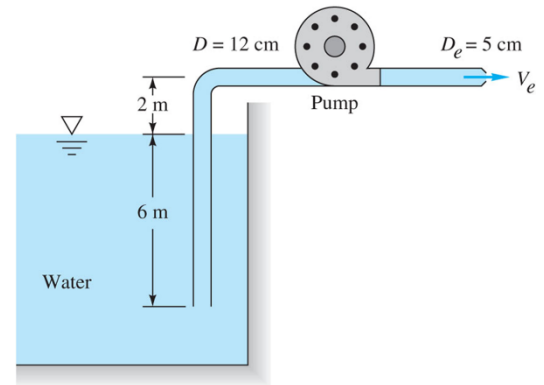
Energy Equation (for incompressible steady flow):

$$\left(\frac{p}{\rho g} + \frac{\alpha V^2}{2g} + z \right)_1 = \left(\frac{p}{\rho g} + \frac{\alpha V^2}{2g} + z \right)_2 + h_{\text{turbine}} - h_{\text{pump}} + h_{\text{friction}}$$

Turbulent pipe flow kinetic correction factor $\alpha = 1.11$

$$P_{\text{pump}} = \rho g Q h_{\text{pump}}$$

Hint: write the energy equation between the free surface and the nozzle exit.



Solution: **Format (+3)**

Continuity:

$$Q = 220 \frac{m^3}{hr} = \frac{(220)}{(3600)} \frac{m^3}{s} = 0.0611 \frac{m^3}{s}$$

$$Q = A_e V_e$$

$$V_e = \frac{Q}{A_e} = \frac{Q}{\pi \frac{D_e^2}{4}} = \frac{(0.0611)}{\pi \frac{(0.05)^2}{4}} = 31.12 \frac{m}{s} \quad (+2)$$

Take point 1 at the free surface and 2 at the pipe exit:

$$\frac{p_1}{\rho g} + \frac{\alpha_1 V_1^2}{2g} + z_1 = \frac{p_2}{\rho g} + \frac{\alpha_2 V_2^2}{2g} + z_2 + h_{turbine} - h_{pump} + h_{friction}$$

Pressure is zero at the free surface and at the exit to atmosphere. Also assume that velocity at the free surface is almost zero:

$$0 + 0 + z_1 = 0 + \frac{\alpha_e V_e^2}{2g} + z_2 + 0 - h_{pump} + h_{friction}$$

$$h_{pump} = \frac{\alpha_e V_e^2}{2g} + (z_2 - z_1) + h_{friction}$$

$$h_{pump} = \frac{(1.11)(31.12)^2}{2(9.81)} + (2 \text{ m}) + (5 \text{ m}) = 61.79 \text{ m}$$

The pump power is:

$$P_{pump} = \rho g Q h_{pump} = (998)(9.81)(0.0611)(61.79) = 36,962.3 \text{ W} = 36.96 \text{ kW} \quad (+5)$$