

Review for Test 3

Chapter 9 Surface Resistance

- $C_D = C_{D_p} + C_f$
form drag skin-friction drag

- Some exact solutions to the Navier-Stokes equations:
Couette flow (without and with ∇p and incline)
and flow down an inclined plane

- Laminar boundary layer

$$\rho(uu_x + vu_y) = -p_x + \mu u_{yy}$$

$$p = p_e$$

$$u_x + v_y = 0$$

- Blasius solution for laminar flow over a flat plate

- Turbulent boundary layer

(1) Stability and transition

(2) Decomposition of turbulent flows: mean-flow boundary layer equations

$$\rho(uu_x + vu_y) = -p_x + \mu u_{yy} - \frac{\partial}{\partial y}(\overline{u'v'})$$

$$p = p_e$$

$$u_x + v_y = 0$$

(3) Semi theoretical considerations of turbulent flow

(a) Eddy-viscosity theory

$$-\rho \overline{u'v'} = \mu_t \frac{\partial u}{\partial y}$$

$$\mu_t = \rho l^2 \left| \frac{\partial u}{\partial y} \right|$$

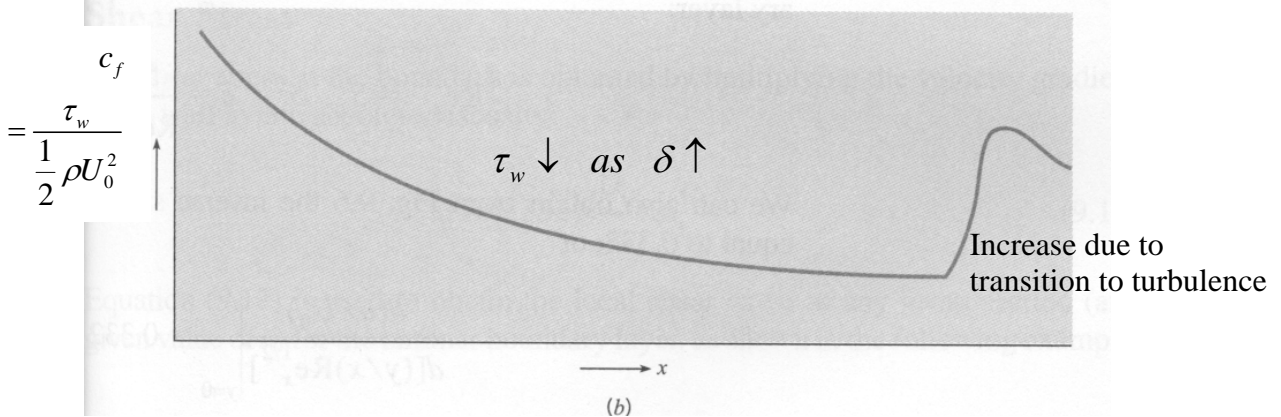
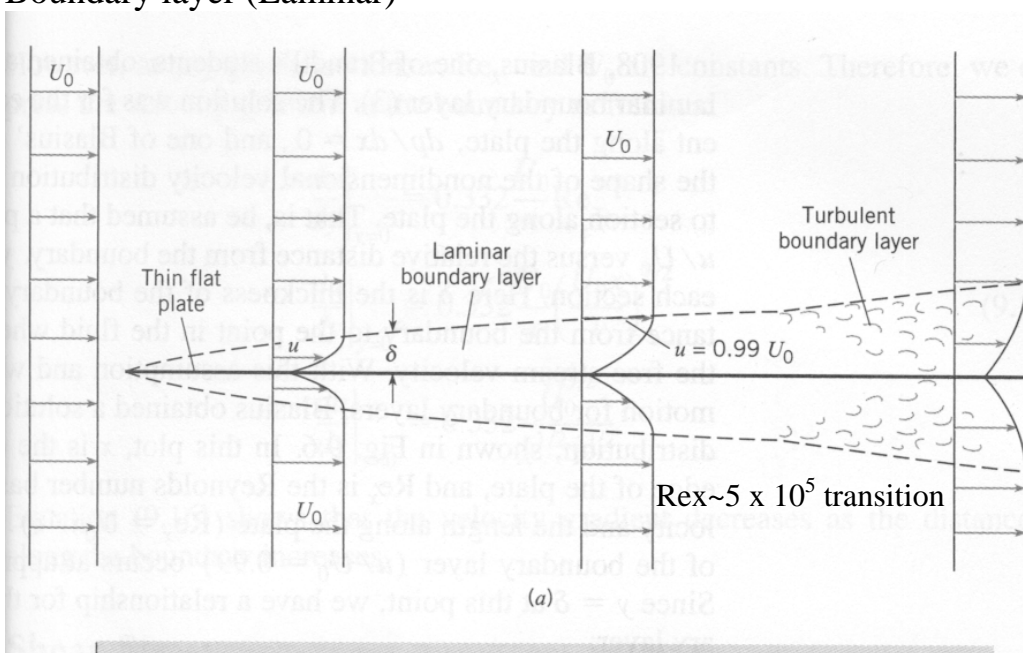
(b) Velocity profile correlations

Inner layer: $u^+ = y^+$

Overlap layer: $u^+ = \frac{1}{\kappa} \ln y^+ + B$

Outer layer: $\frac{U_e - u}{u_*} = f(y/\delta)$

- Boundary layer (Laminar)



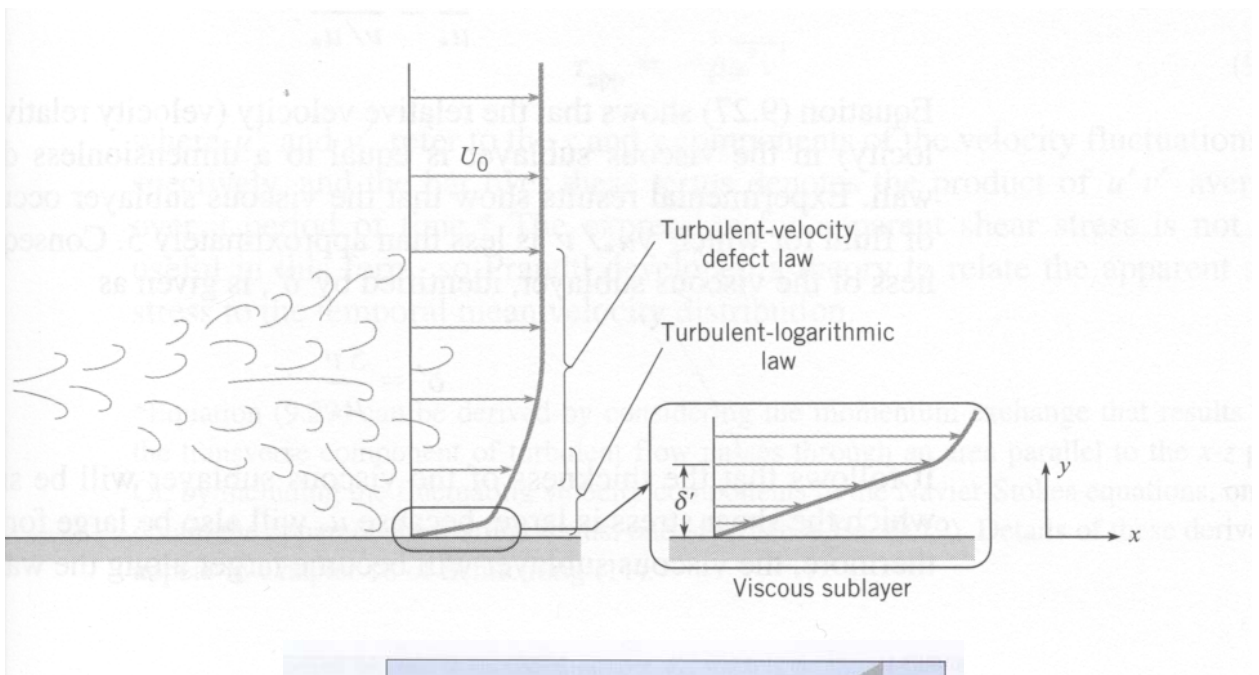
- Laminar flow solution by Blasius (1908):

$$\delta/x = 5/\sqrt{\text{Re}_x}$$

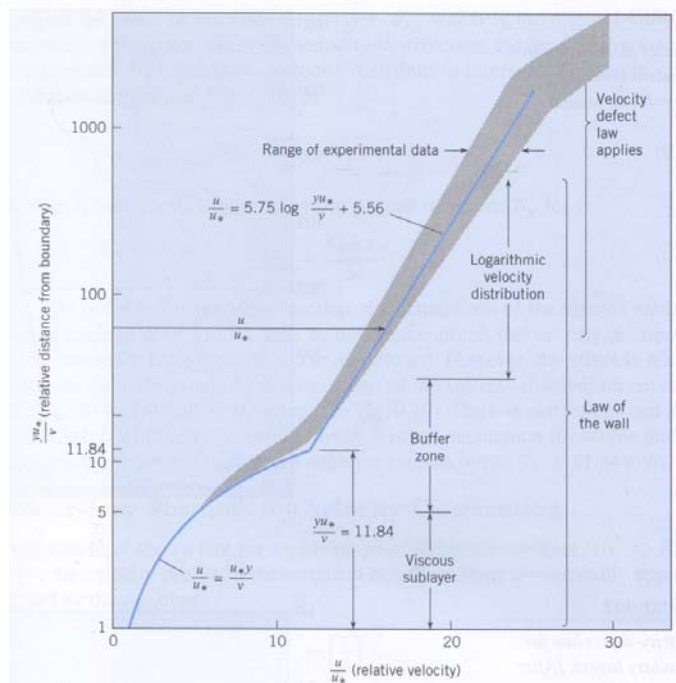
$$c_f = 0.664/\sqrt{\text{Re}_x} \text{ local wall-shear stress coefficient}$$

$$C_f = B \int_0^L c_f dx = 1.33/\sqrt{\text{Re}_L} \text{ shear stress coefficient} = \frac{F_s}{\frac{1}{2}\rho U_0^2 BL}$$

- Boundary layer (turbulent)



$$\frac{u}{U_0} \sim \left(\frac{y}{\delta}\right)^{1/7}$$



- Application of the momentum equation yields

$$\tau_w = \rho U_0^2 \frac{d\theta}{dx}$$

$$\text{i.e., } \frac{c_f}{2} = \frac{d\theta}{dx} \text{ where } \theta = \text{momentum thickness} = \int_0^\delta \frac{u}{U_0} \left(1 - \frac{u}{U_0}\right) dy$$

= loss of momentum due to boundary layer

$$\text{For } \frac{u}{U_0} = \left(\frac{y}{\delta}\right)^{1/7}, \text{ Re} < 10^7$$

$$\Rightarrow \theta = \frac{2}{72} \delta = \theta(\delta)$$

$$\frac{c_f}{2} = \frac{d\theta}{dx} \Rightarrow \frac{\tau_w}{\rho} = \frac{2}{72} U_0^2 \frac{d\delta}{dx}$$

From power-law fit to log-law evaluated at δ

$$\Rightarrow \frac{u}{u^*} = 8.74 \left(\frac{\delta u^*}{\nu}\right)^{1/7} \quad u^* = \sqrt{\tau_w / \rho}$$

Combining these equations and integrating yields

$$\text{Boundary layer thickness } \frac{\delta}{x} = 0.37 / \text{Re}_x^{1/5}$$

$$\delta \propto x^{4/5}$$

$$\text{Laminar } \delta \propto x^{1/2}$$

Faster growth than laminar, almost linear

$$\text{Local shear friction: } c_f = \frac{\tau_w}{\rho U_0^2 / 2} = \frac{0.058}{\text{Re}_x^{1/5}}$$

$$\text{Skin friction drag: } C_f = \frac{1}{L} \int_0^L c_f dx = \frac{F_s}{(\rho U_0^2 / 2) BL} = \frac{0.074}{\text{Re}_L^{1/5}}$$

See text for some higher Re formulas

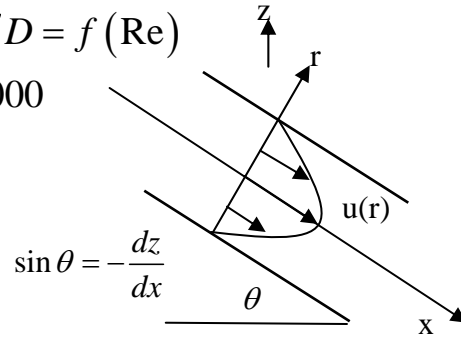
Chapter 10 Flow in Conduit

- Entrance vs. fully developed flow: $L_e/D = f(\text{Re})$
- Laminar vs. turbulent flow: $\text{Re}_{crit} \approx 2000$

Summary: Laminar pipe flow solution

$$u(r) = \frac{r^2 - r_0^2}{4\mu} \frac{d}{dx}(p + \gamma z),$$

$$u(r) > 0 \text{ for } \frac{d}{dx}(p + \gamma z) < 0, \text{ i.e., favorable pressure gradient}$$



$$\tau = -\frac{r}{2} \frac{d}{dx}(p + \gamma z)$$

$$\tau_0 = -\frac{r_0}{2} \frac{d}{dx}(p + \gamma z)$$

$$V = \frac{Q}{A} = -\frac{r_0^2}{8\mu} \frac{d}{dx}(p + \gamma z)$$

$$V = \frac{-D^2}{32\mu} \frac{[p_2 + \gamma z_2 - p_1 - \gamma z_1]}{L}$$

$$\frac{p_1}{\gamma} + z_1 + \frac{V^2}{2g} = \frac{p_2}{\gamma} + z_2 + \frac{V^2}{2g} + h_L$$

$$-\gamma h_L = (p_2 + \gamma z_2) - (p_1 + \gamma z_1) \Rightarrow \frac{-\gamma h_L}{L} = \frac{d}{dx}(p + \gamma z)$$

$$h_L = \frac{32\mu LV}{\gamma D^2} = f \frac{L V^2}{D 2g}$$

$$\tau_0 = \frac{D \gamma h_L}{4 L} \Rightarrow h_L = \frac{4L\tau_0}{D\gamma} = 4L \frac{1}{8} \rho V^2 f / D \gamma = f \frac{L V^2}{D 2g}$$

$$\therefore f = \frac{64}{\text{Re}} \quad \text{Re} = \frac{VD}{\nu}$$

In general $f = f(\text{Re}, k_s/D)$, $k_s/D = \text{roughness}$

Summary: Turbulent pipe flow

Friction factor:

$$1/\sqrt{f} = 1.14 - 2 \log(k_s/D + 9.35/\text{Re}\sqrt{f}), \text{ as shown in Moody diagram}$$

$k_s/D =$ roughness parameter

$$h_L = f \frac{L V^2}{D 2g} = h_1 - h_2, \quad h_1 = \frac{p_1}{\gamma} + z_1, \quad h_2 = \frac{p_2}{\gamma} + z_2$$

Types of problems:

1. Determine head loss

$$f = f(\text{Re}, k_s/D)$$

$$h_L = f \frac{L V^2}{D 2g} = h_1 - h_2$$

2. Determine flow rate

$$V = \underbrace{\left[\frac{2gh_L}{LD} \right]}_{\text{known from data}}^{1/2} f^{-1/2}$$

guess $f \Rightarrow V \Rightarrow \text{Re} \Rightarrow f$, repeat to converge

3. Determine pipe size

$$D = \underbrace{\left[\frac{8LQ^2}{\pi^2 gh_L} \right]}_{\text{known from data}}^{1/5} f^{1/5}$$

guess $f \Rightarrow D \Rightarrow \text{Re} \& k_s/D \Rightarrow f$, repeat to converge

Alternate method is experiment formulas in text

$$f = \frac{0.25}{\left[\log \left(\frac{k_s}{3.7D} + \frac{5.74}{\text{Re}^{0.9}} \right) \right]^2}$$

$$Q = -2.22D^{5/2} \sqrt{gh_L/L} \log \left(\frac{k_s}{3.7D} + \frac{1.78v}{D^{3/2} \sqrt{gh_L/L}} \right)$$

$$D = 0.66 \left[k_s^{1.25} \left(\frac{LQ^2}{gh_L} \right)^{4.75} + vQ^{9.4} \left(\frac{L}{gh_L} \right)^{5.2} \right]^{0.04}$$

Minor losses:

$$\frac{p_1}{\gamma} + z_1 + \alpha_1 \frac{V_1^2}{2g} + h_p = \frac{p_2}{\gamma} + z_2 + \alpha_2 \frac{V_2^2}{2g} + h_t + h_L + \sum_i h_{m_i}$$

$$h_{m_i} = \left(K \frac{V^2}{2g} \right)_i$$

Chapter 11 External flows

- $$F_D = \int_A (-p \cos \theta + \tau \sin \theta) dA$$

$$dF_D = (-p \mathbf{n} + \tau \mathbf{t}) \cdot \mathbf{t} dA$$

$$C_D = \frac{F_D}{\frac{1}{2} \rho U_0^2 A_p} = C_{D_{Form}} + C_{D_{Skin Friction}}$$

Often must be obtained experimentally

Projection

$$C_D = C_D \left(\text{geometry}, \text{Re}, \frac{k_s}{L}, \text{Ma}, \text{etc.} \right)$$

- Influence of geometry:
 - angular vs. curved bodies
 - streamlining
 - $C_{D_{Form}}$ vs. $C_{D_{SF}}$ as $f(\text{geometry})$

- Roughness

- Vortex shedding: Strouhal No. = $S = \frac{nd}{U_0} = S(\text{Re})$

- Terminal velocity: $\sum \mathbf{F} = 0 \Rightarrow \mathbf{a} = 0$ define V_{TERMIN}

- $C_D(\text{Ma})$ use near $\text{Ma} \sim 1$ due to shock waves

- Lift

