

Prob. 1**Information and assumptions**

Provided in problem statement

Find

Determine pressure just upstream of the nozzle.

Solution

Bernoulli equation:

$$\frac{p_1}{\gamma} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\gamma} + \frac{V_2^2}{2g} + z_2 \quad (+6)$$

With:

$$z_1 = z_2, \quad p_2 = 0 \quad (+1)$$

and:

$$Q = 250 \times 0.002228 = 0.557 \text{ ft}^3/\text{s}$$

$$V_1 = \frac{Q}{A_1} = \frac{0.557}{\frac{\pi}{4} \left(\frac{1.125}{12} \right)^2} = 80.7 \text{ ft/s} \quad (+1)$$

$$V_2 = \frac{Q}{A_2} = \frac{0.557}{\frac{\pi}{4} \left(\frac{3}{12} \right)^2} = 11.3 \text{ ft/s} \quad (+1)$$

Then:

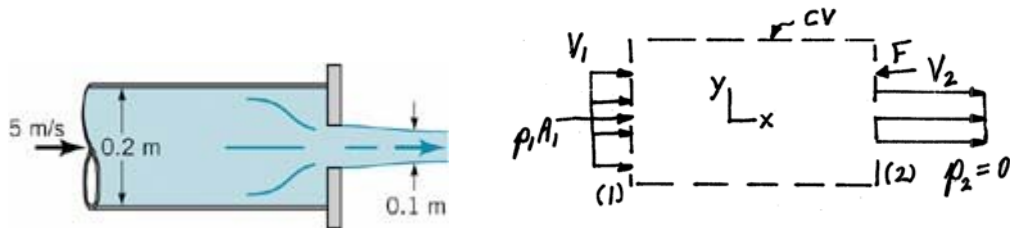
$$\begin{aligned} p_1 &= \frac{\gamma}{2g} (V_2^2 - V_1^2) = \frac{\rho}{2} (V_2^2 - V_1^2) \\ &= \frac{1}{2} (1.94) (80.7^2 - 11.3^2) = 6192 \text{ lbf/ft}^2 \quad (+1) \end{aligned}$$

Prob. 2**Information and assumptions**

Provided in problem statement

Find

Determine the force needed to hold the plate against the pipe.

Solution

The x-component of the momentum equation for the control volume shown is:

$$\int u \rho \mathbf{V} \cdot \mathbf{n} dA = \sum F_x$$

Or

$$V_1 \rho (-V_1) A_1 + V_2 \rho (V_2) A_2 = p_1 A_1 - F \quad (+6)$$

Thus

$$F = p_1 A_1 + \rho V_1^2 A_1 - \rho V_2^2 A_2 = p_1 A_1 + \dot{m} (V_1 - V_2) \quad (+1)$$

Where: $\dot{m} = \rho V_1 A_1 = 999 \times \frac{\pi}{4} \times 0.2^2 \times 5 = 157 \text{ kg/s}$

Continuity gives $V_1 A_1 = V_2 A_2$

$$V_2 = \frac{A_1}{A_2} V_1 = \left(\frac{d_1}{d_2} \right)^2 V_1 = \left(\frac{0.2}{0.1} \right)^2 \times 5 = 20 \text{ m/s} \quad (+1)$$

In addition Bernoulli equation gives $p_1 + \frac{1}{2} \rho V_1^2 = p_2 + \frac{1}{2} \rho V_2^2$ with $p_2 = 0$

$$p_1 = \frac{\rho}{2} (V_2^2 - V_1^2) = \frac{1}{2} \times 999 \times (20^2 - 5^2) = 1.87 \times 10^5 \text{ N/m}^2 \quad (+1)$$

Therefore

$$F = 1.87 \times 10^5 \times \frac{\pi}{4} \times 0.2^2 + 157 \times (5 - 20) = 3520 \text{ N} \quad (+1)$$

Prob. 3**Information and assumptions**

Provided in problem statement

Find

Corresponding velocity and wave resistance of the prototype .

Solution

Dynamic similarity by equating Froude numbers:

$$Fr_m = Fr_p \quad (+4)$$

$$\frac{V_m}{\sqrt{gL_m}} = \frac{V_p}{\sqrt{gL_p}}$$

$$V_p = \sqrt{\frac{gL_p}{gL_m}} V_m = \sqrt{\frac{L_p}{L_m}} V_m = \sqrt{\frac{16}{1}} \times 3 = 12 \text{ m/s} \quad (+2)$$

$$\frac{F_p}{F_m} = \left(\frac{L_p}{L_m} \right)^3 \quad (+3)$$

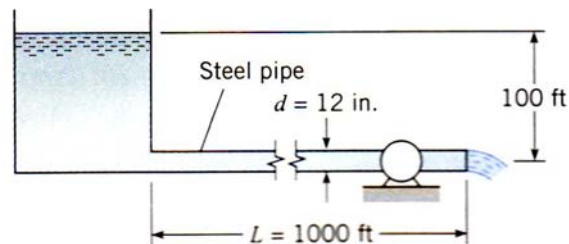
$$F_p = \left(\frac{L_p}{L_m} \right)^3 F_m = \left(\frac{16}{1} \right)^3 \times 12 = 49152 \text{ N} \quad (+1)$$

Prob. 4**Information and assumptions**

Provided in problem statement

Find

Power delivered by the turbine

Solution

Energy equation from the reservoir water surface to the jet at the end of the pipe:

$$\frac{p_1}{\gamma} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\gamma} + \frac{V_2^2}{2g} + z_2 + h_T + \sum h_L$$

$$0 + 0 + z_1 = 0 + \frac{V^2}{2g} + z_2 + h_T + \left(K_e + f \frac{L}{D} \right) \frac{V^2}{2g} \quad (+3)$$

$$z_1 - z_2 = h_T + \left(1 + 0.5 + f \frac{L}{D} \right) \frac{V^2}{2g} \quad (+1)$$

But:

$$V = \frac{Q}{A} = \frac{5}{\frac{\pi}{4} \times 1^2} = 6.37 \text{ ft/s}$$

$$\frac{V^2}{2g} = 0.630 \text{ ft}$$

$$\text{Re} = \frac{VD}{\nu} = \frac{6.37 \times 1}{1.21 \times 10^{-5}} = 5.26 \times 10^5 \quad (+2)$$

From Moody diagram $f = 0.0153$ for $k_s/D = 0.00015$ (+2)

$$100 = h_T + \left(1 + 0.5 + 0.0153 \times \frac{1000}{1} \right) \times 0.629$$

$$h_T = 89.4 \text{ ft} \quad (+1)$$

So power delivered by the turbine

$$p = \eta Q \gamma h_T = 80\% \times 5 \times 62.4 \times 89.4 = 22314 \text{ ft} \cdot \text{lb} / \text{s} = 40.57 \text{ hp} \quad (+1)$$

Prob. 5**Information and assumptions**

Provided in problem statement

Find

Determine the boundary layer thickness

Solution

Calculate the Reynolds number at the end of the plate:

$$\text{Re}_x = \frac{Vx}{\nu} = \frac{15.5 \times 10.6}{1.57 \times 10^{-4}} = 1.05 \times 10^6 > 5 \times 10^5 \quad (+3)$$

So the boundary layer is transitional. (+1)

Suppose the boundary layer is laminar everywhere

$$\delta = \frac{5x}{\sqrt{\text{Re}_x}} = \frac{5 \times 10.6}{\sqrt{1.06 \times 10^6}} = 0.0515 \text{ ft} = 0.618 \text{ in} \quad (+3)$$

Suppose the boundary layer is turbulent everywhere

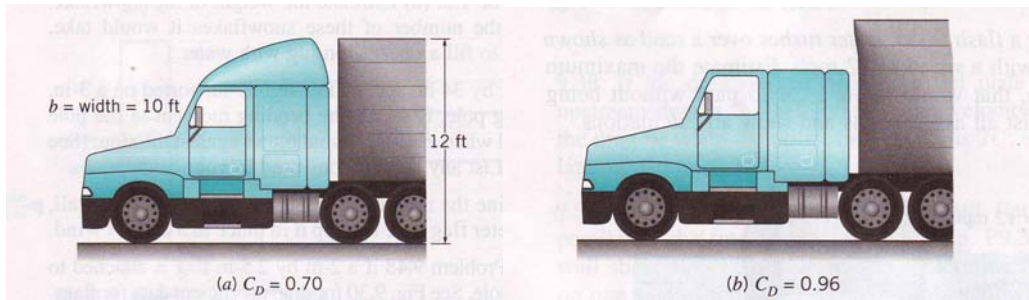
$$\delta = \frac{0.16x}{(\text{Re}_x)^{1/7}} = \frac{0.16 \times 10.6}{(1.06 \times 10^6)^{1/7}} = 0.234 \text{ ft} = 2.80 \text{ in} \quad (+3)$$

Prob. 6**Information and assumptions**

Provided in problem statement

Find

Determine a reduction of how many horsepower needed at a highway speed of 65 mph.

Solution

Drag:

$$D = C_D \frac{1}{2} \rho U^2 A \quad (+4)$$

Drag reduction:

$$\Delta D = (C_{Db} - C_{Da}) \frac{1}{2} \rho U^2 A \quad (+3)$$

Power

$$P = DU \quad (+1)$$

With $U = 65 \text{ mph} = 95.3 \text{ ft/s}$

Power reduction:

$$\begin{aligned} \Delta P &= (\Delta D)U = (C_{Db} - C_{Da}) \frac{1}{2} \rho U^3 A \\ &= (0.96 - 0.70) \times \frac{1}{2} \times 0.00238 \times 95.3^3 \times 10 \times 12 \\ &= 32135 \text{ ft} \cdot \text{lb/s} \\ &= 58.4 \text{ hp} \quad (+2) \end{aligned}$$