

Prob. 1**Information and assumptions****Find**

Is the magnitude of the shear stress greater at the moving plate or at the stationary plate?

Solution

Develop an equation for the shear stress

$$\tau = \mu \frac{du}{dy} \quad (\text{Eqn.} = 7)$$

$$= -\mu \left(\frac{1}{2\mu} \right) \left(\frac{dp}{ds} \right) (H - 2y) + \mu \frac{u_t}{H}$$

Evaluate τ at $y = H$:

$$\tau_H = \frac{1}{2} \left(\frac{dp}{ds} \right) H + \mu \frac{u_t}{H}$$

Evaluate τ at $y = 0$:

$$\tau_0 = -\frac{1}{2} \left(\frac{dp}{ds} \right) H + \mu \frac{u_t}{H} \quad (\text{Inter.} = 2)$$

Observation of the velocity gradient lets one conclude that the pressure gradient dp/ds is negative. Also u_t is negative. Therefore:

$$|\tau_H| > |\tau_0| \quad (\text{Ans.} = 1)$$

Prob. 2**Information and assumptions**

Provided in problem statement

Find

Force at flange to hold nozzle in place

Solution

Velocity calculation

$$v_1 = \frac{Q}{A_1} = \frac{15}{(\pi/4) \times 1^2} = 19.10 \text{ ft/s}$$

$$v_2 = \frac{Q}{A_2} = \frac{15}{(\pi/4) \times (8/12)^2} = 42.97 \text{ ft/s} \quad (\text{Eqn. + Ans.} = 2)$$

Bernoulli equation

$$p_1 + \frac{1}{2} \rho v_1^2 = p_2 + \frac{1}{2} \rho v_2^2 \quad (\text{Eqn.} = 2)$$

$$\begin{aligned} p_1 &= 0 + \frac{1}{2} \rho (v_2^2 - v_1^2) \\ &= 1,437 \text{ lbf} / \text{ft}^2 \quad (\text{Inter. + Ans.} = 1) \end{aligned}$$

x-momentum equation

$$\sum F_x = \dot{m}_2 v_2 - \dot{m}_1 v_1 \quad (\text{Eqn.} = 4)$$

$$p_1 A_1 + F = (1.94 \times 15) \times 42.97 - (1.94 \times 15) \times 19.10$$

$$1,437 \times \left(\frac{\pi}{4} \right) \times 1^2 + F = (1.94 \times 15) \times (42.97 - 19.10)$$

$$F = -434 \text{ lbf} \text{ (act to left)} \quad (\text{Inter. + Ans.} = 1)$$

Prob. 3**Information and assumptions**

Provided in problem statement

From Table A.3 $\nu = 1.5 \times 10^{-5} \text{ m}^2/\text{s}$ and $\rho = 1.2 \text{ kg}/\text{m}^3$

Find

Total drag force on plate

Solution

The force due to shear stress is

$$F_s = C_f \frac{1}{2} \rho U_0^2 BL \quad (\text{Eqn.} = 6)$$

The Reynolds number based on the plate length is

$$\text{Re}_L = \frac{U_0 L}{\nu} = \frac{15 \times 1}{1.5 \times 10^{-5}} = 10^6 \quad (\text{Inter.} + \text{Ans.} = 1)$$

The average shear stress coefficient on the tripped side of the plate is

$$C_f = \frac{0.074}{(10^6)^{1/5}} = 0.0047 \quad (\text{Inter.} + \text{Ans.} = 2)$$

The total force is

$$F_x = 0.0047 \times \left(\frac{1}{2} \times 1.2 \times 15^2 \right) \times (1.0 \times 0.5) = 0.317 \text{ N} \quad (\text{Inter.} + \text{Ans.} = 1)$$

Prob. 4

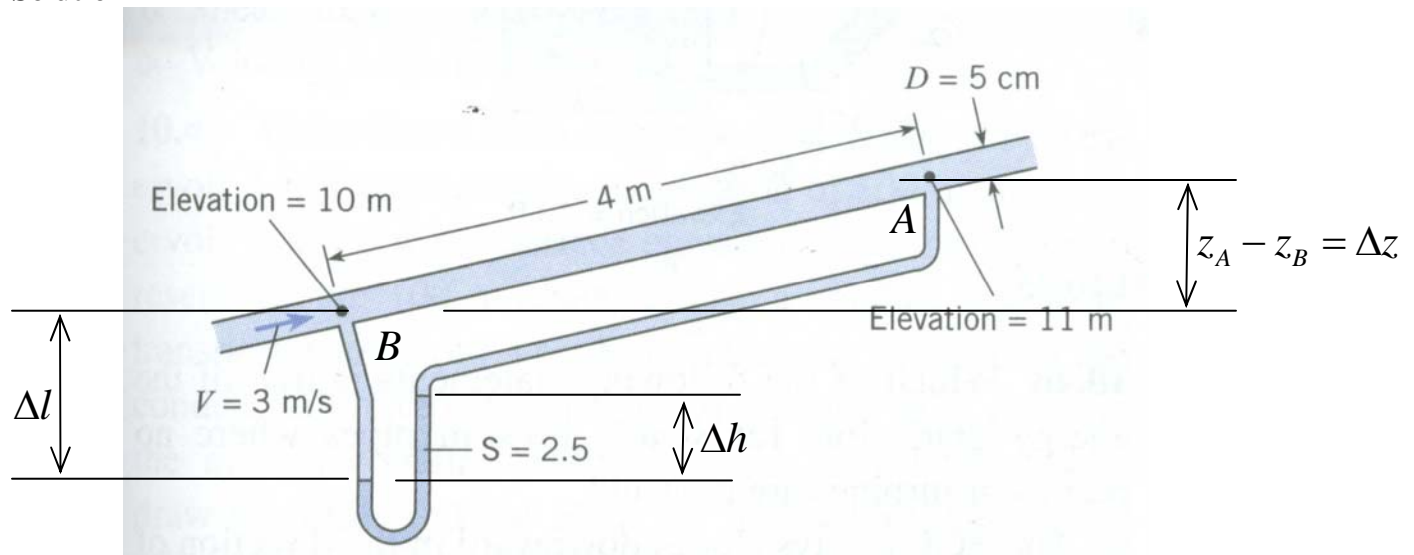
Information and assumptions

Provided in problem statement

Find

Resistance coefficient

Solution



Manometer equation

$$p_B + \gamma_f \Delta l - \gamma_m \Delta h - \gamma_f [(\Delta l - \Delta h) + \Delta z] = p_A$$

$$p_B - \gamma_m \Delta h - \gamma_f [-\Delta h + (z_A - z_B)] = p_A$$

$$p_B - (\gamma_m - \gamma_f) \Delta h - \gamma_f z_A + \gamma_f z_B = p_A$$

$$(p_B + \gamma_f z_B) - (p_A + \gamma_f z_A) = (\gamma_m - \gamma_f) \Delta h$$

$$p_{zB} - p_{zA} = (\gamma_m - \gamma_f) \Delta h$$

$$\Delta h = \frac{p_{zB} - p_{zA}}{\gamma_f} = \frac{(\gamma_m - \gamma_f) \Delta h}{\gamma_f} = \left(\frac{\gamma_m}{\gamma_f} - 1 \right) \Delta h = (2.5 - 1) \Delta h$$

$$\Delta h = h_f = 0.80 \times (2.5 - 1) = 1.2 \text{ m}$$

$$h_f = f \frac{L V^2}{D 2g}$$

$$f = 1.2 \times \left(\frac{0.05}{4} \right) \times \frac{2 \times 9.81}{3^3} = 0.033$$

(Eqn + Inter. + Ans. = 3)

(Eqn = 6)

(Inter. + Ans. = 1)

Prob. 5**Information and assumptions**

Provided in problem statement

From Table 10.3: $K_e = 0.03$; $K_b = 0.35$; $K_E = 1.0$

From Table A.5, $\nu = 10^{-6} \text{ m}^2/\text{s}$

From Table 10.2, $k_s = 0.046 \text{ mm}$

Find

The pump power

Solution

Energy equation from the water surface in the lower reservoir to the water surface in the upper reservoir

$$p_1/\gamma + V_1^2/2g + z_1 + h_p = p_2/\gamma + V_2^2/2g + z_2 + \sum h_L$$

$$0 + 0 + 200\text{m} + h_p = 0 + 0 + 235\text{m} + \frac{V_2^2}{2g} \left(K_e + K_b + K_E + f \frac{L}{D} \right)$$

(Eqn = 7)

$$V = \frac{Q}{A} = \frac{0.314}{\left(\frac{\pi}{4}\right) \times 0.3^2} = 4.442 \text{ m/s}, \quad \frac{V^2}{2g} = 1.01\text{m}$$

$$\text{Re} = \frac{VD}{\nu} = \frac{4.44 \times 0.3}{10^{-6}} = 1.33 \times 10^6, \quad \frac{k_s}{\nu} \approx 0.00015$$

So from Figure 10.8 $f = 0.014$

$$f \frac{L}{D} = 0.014 \times \frac{140}{0.3} = 6.53$$

(Inter.+ Ans. = 2)

$$h_p = 235 - 200 + 1.01 \times (0.03 + 0.35 + 1 + 6.53) = 43.0\text{m}$$

$$p = Q\gamma h_p = 0.314 \times 9,790 \times 43.0 = 132\text{kW}$$

(Inter.+ Ans. = 1)

Prob. 6**Information and assumptions**

Provided in problem statement

From Table A.3, $\rho = 0.00237 \text{ slugs}/\text{ft}^3$; $\nu = 1.58 \times 10^{-4} \text{ ft}^2/\text{s}$

Find

Drag on cooling tower

Solution

$$V = 4.31 \text{ mph} = 6.32 \text{ ft/s}$$

$$\text{Re} = \frac{V_0 d}{\nu} = \frac{6.32 \times 250}{1.58 \times 10^{-4}} = 1.0 \times 10^7$$

From Figure 11.5: $C_D = 0.70$ (Eqn. = 2)

Then

$$F_D = C_D A_p \frac{1}{2} \rho V^2 \quad (\text{Eqn.} = 7)$$

$$= 0.70 \times 250 \times 350 \times \frac{1}{2} \times 0.00237 \times 6.32^2$$

$$= 2,899 \text{ lbf} \quad (\text{Inter.} + \text{Ans.} = 1)$$
