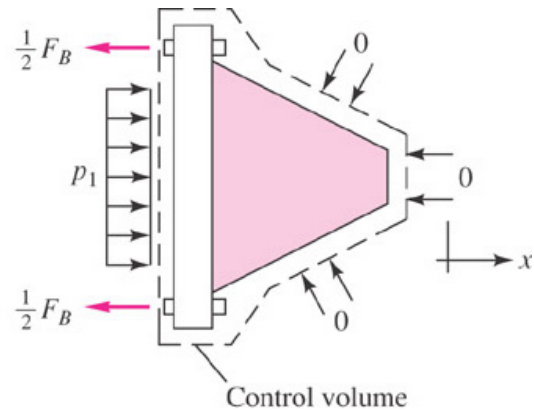


EXAM2 Solutions**Problem 1: Momentum equation (Chapter 5)****Information and assumptions**

- Frictionless flow
- $D_1 = 10 \text{ cm} = 0.1 \text{ m}$
- $D_2 = 3 \text{ cm} = 0.03 \text{ m}$
- $Q = 1.5 \text{ m}^3/\text{min} = 0.025 \text{ kg/s}$
- $\rho_{\text{water}} = 1000 \text{ kg/m}^3$
- $p_2 = 0$ (gage)

Find

- The force F_B exerted by the flange bolts

**Solution**

x-momentum equation:

$$-F_B + p_1 A_1 = \dot{m}(V_2 - V_1)$$

where,

$$\dot{m} = \rho Q = 1000 \text{ kg/m}^3 \times (0.025 \text{ m}^3/\text{s}) = 25 \text{ kg/s}$$

$$V_1 = \frac{Q}{A_1} = \frac{0.025 \text{ m}^3/\text{s}}{\pi(0.1 \text{ m})^2/4} = 3.18 \text{ m/s}$$

$$V_2 = \frac{Q}{A_2} = \frac{0.025 \text{ m}^3/\text{s}}{\pi(0.03 \text{ m})^2/4} = 35.37 \text{ m/s} \quad (+5 \text{ points})$$

p_1 can be determined by using the Bernoulli equation:

$$p_1 + \frac{1}{2}\rho V_1^2 + z_1 = p_2 + \frac{1}{2}\rho V_2^2 + z_2$$

Since the hose is horizontal, $z_1 = z_2$ and $p_2 = p_{\text{atm}} = 0$ gage

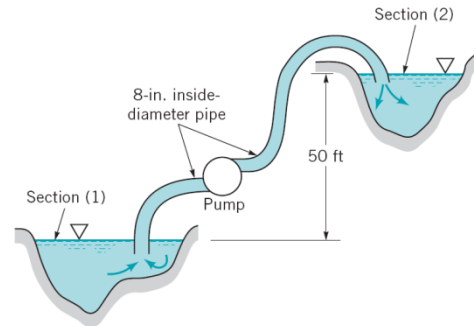
$$\begin{aligned} p_1 &= \frac{1}{2}\rho(V_2^2 - V_1^2) \\ &= \frac{1}{2} \times 1000 \text{ kg/m}^3 \times [(35.37 \text{ m/s})^2 - (3.18 \text{ m/s})^2] = 620,462 \text{ Pa} \quad (+4 \text{ points}) \end{aligned}$$

Thus,

$$\begin{aligned} F_B &= p_1 A_1 - \dot{m}(V_2 - V_1) \\ &= 620462 \text{ N/m}^2 \times \frac{\pi(0.1 \text{ m})^2}{4} - 25 \text{ kg/s} \times (35.37 \text{ m/s} - 3.18 \text{ m/s}) \\ &= \mathbf{4068 \text{ N}} \quad (+1 \text{ point}) \end{aligned}$$

EXAM2 Solutions**Problem 2: Energy equation (Chapter 5)****Information and assumptions**

- $Q = 2.5 \text{ ft}^3/\text{s}$
- $\text{loss} = 61\bar{V}^2/2$
- $z_2 - z_1 = 50 \text{ ft}$
- $D = 8 \text{ in}$
- $p_1 = p_2 = p_{atm}$
- The reservoirs are large enough $\Rightarrow V_1 \approx V_2 \approx 0$

**Find**

- Shaft power \dot{W}_s

Solution

Energy equation:

$$\frac{p_2}{\rho} + \frac{V_2^2}{2} + gz_2 = \frac{p_1}{\rho} + \frac{V_1^2}{2} + gz_1 + w_s - \text{loss}$$

where, $p_1 = p_2 = p_{atm} = 0$ (gage) and $V_1 \approx V_2 \approx 0$ since the reservoirs are large enough.

The required shaft power is

$$\dot{W}_s = \dot{m}w_s = \dot{m}[g(z_2 - z_1) + \text{loss}] \quad (+6 \text{ points})$$

where,

$$\dot{m} = \rho Q = 1.94 \text{ slugs/ft}^3 \times 2.5 \text{ ft}^3/\text{s} = 4.85 \text{ slugs/s}$$

$$\bar{V} = Q/A = \frac{4Q}{\pi D^2} = \frac{4 \times 2.5 \text{ ft}^3/\text{s}}{\pi \times (8/12 \text{ ft})^2} = 7.162 \text{ ft/s}$$

$$\text{loss} = 61\bar{V}^2/2 = 61 \times \frac{(7.162 \text{ ft/s})^2}{2} = 1564.5 \text{ ft}^2/\text{s}^2 \quad (+3 \text{ points})$$

or

$$\dot{W}_s = 4.85 \text{ slugs/s} \times [32.2 \text{ ft/s}^2 \times 50 \text{ ft} + 1564.5 \text{ ft}^2/\text{s}^2] = 15396 \text{ slugs} \cdot \text{ft}^2/\text{s}^3$$

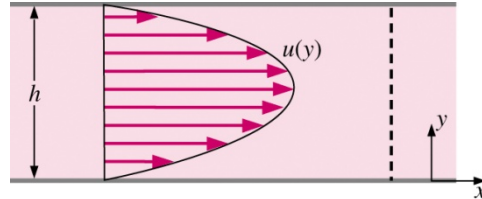
$$= 15396 \text{ slugs} \cdot \text{ft}^2/\text{s}^3 \times \frac{1 \text{ hp}}{550 \text{ slugs} \cdot \text{ft}^2/\text{s}^3} = \mathbf{28 \text{ hp}} \quad (+1 \text{ points})$$

EXAM2 Solutions

Problem 3: Bernoulli equation and manometer (Chapter 3)

Information and assumptions

- Steady ($\partial/\partial t = 0$), fully developed ($\partial/\partial x = 0$)
- 2-D flow ($w = 0$, $\partial/\partial z = 0$)
- $\mu = 0.38 \text{ N} \cdot \text{s}/\text{m}^2$
- $h = 0.004 \text{ m}$
- $\Delta p/\ell = -dp/dx = 20,000 \text{ Pa}/\text{m}$



Find

- Velocity distribution
- Maximum velocity u_{max}

Solution

Since flow is steady, fully developed, 2-D, the Navier-Stokes equations reduce to

$$\mu \frac{d^2 u}{dy^2} = -\frac{dp}{dx} \quad (+4 \text{ points})$$

By integrating the equation twice with respect to y and using boundary conditions

$$u(y) = \frac{1}{2\mu} \left(\frac{dp}{dx} \right) y^2 + C_1 y + C_2$$

$$\text{B.C. } u(0) = C_2 = 0$$

$$u(h) = \frac{1}{2\mu} \left(\frac{dp}{dx} \right) h^2 + C_1 h = 0 \quad \Rightarrow C_1 = -\frac{1}{2\mu} \left(\frac{dp}{dx} \right) h, C_2 = 0$$

Thus the velocity profile is

$$u(y) = \frac{1}{2\mu} \left(\frac{dp}{dx} \right) y(y - h) \quad (+3 \text{ points})$$

The maximum velocity occurs $y = \frac{h}{2}$

$$u_{max} = \frac{1}{2\mu} \left(\frac{dp}{dx} \right) \frac{h}{2} \left(\frac{h}{2} - h \right) = -\frac{h^2}{8\mu} \left(\frac{dp}{dx} \right) \quad (+2 \text{ points})$$

For the oil (SAE 30),

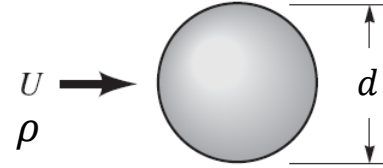
$$u_{max} = -\frac{(0.004 \text{ m})^2}{8 \times 0.38 \text{ N} \cdot \text{s}/\text{m}^2} \times (-20000 \text{ N}/\text{m}^3) = \mathbf{0.105 \text{ m/s}} \quad (+1 \text{ point})$$

EXAM2 Solutions**Problem 4: Dimensional analysis (Chapter 7)****Information and assumptions**

- $D = f(\rho, U, d)$

Find

- A relationship for D as a function of ρ, U, d

**Solution**

From the statement of the problem

$$D = f(\rho, U, d)$$

And the dimensions of the variables are

$$D \doteq MLT^{-2}$$

$$\rho \doteq ML^{-3}$$

$$U \doteq LT^{-1}$$

$$d \doteq L$$

Number of pi terms = $k - r = 4 - 3 = 1$

$$\Pi = D\rho^a U^b d^c \quad (+6 \text{ points})$$

It follows that

$$(MLT^{-2})(ML^{-3})^a(LT^{-1})^b(L)^c = M^0L^0T^0$$

and

$$\begin{aligned} 1 + a &= 0 && \text{(for } M\text{)} \\ 1 - 3a + b + c &= 0 && \text{(for } L\text{)} \\ -2 - b &= 0 && \text{(for } T\text{)} \end{aligned}$$

so that $a = -1, b = -2, c = -2$, and therefore (+3 points)

$$\Pi = \frac{D}{\rho U^2 d^2}$$

Thus, the drag will increase by a **factor of 4** if U is doubled. (+1 points)