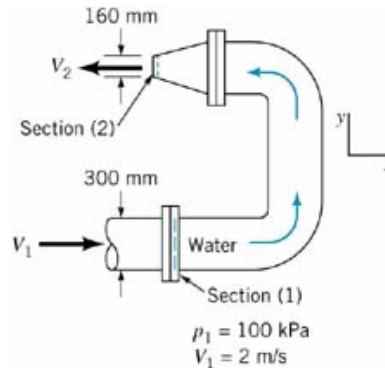


**Prob. 1****Information and assumptions**

Provided in problem statement

**Find**

Determine the magnitude and direction of the anchoring force.

**Solution**

The control volume shown in the sketch above is used.

Application of the x direction linear momentum equation leads to:

$$-u_1 \rho u_1 A_1 - u_2 \rho u_2 A_2 = p_1 A_1 - R_x + p_2 A_2 \quad (+5)$$

From the conservation of mass equation:

$$\dot{m} = \rho u_1 A_1 = \rho u_2 A_2 \quad (+1)$$

$$u_2 = \left( \frac{A_1}{A_2} \right) u_1 = \frac{D_1^2}{D_2^2} u_1 \quad (+1)$$

Then:

$$R_x = \rho u_1 A_1 (u_1 + u_2) + p_1 A_1 + p_2 A_2 \quad (+1)$$

$$= \left[ \rho u_1 \left( u_1 + \frac{D_1^2}{D_2^2} u_1 \right) + p_1 \right] A_1 + (0) A_2$$

$$= \left[ \rho u_1 \left( u_1 + \frac{D_1^2}{D_2^2} u_1 \right) + p_1 \right] \left( \frac{1}{4} \pi D_1^2 \right)$$

$$= \left[ (999)(2) \left( 2 + \frac{0.3^2}{0.16^2} 2 \right) + 100000 \right] \left( \frac{1}{4} \pi \times 0.3^2 \right) = 8344 \text{ N} \quad (+1)$$

Application of the y direction linear momentum equation yields:

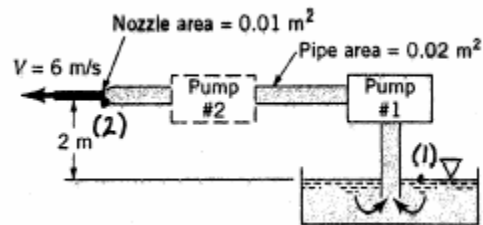
$$R_y = 0 \quad (+1)$$

**Prob. 2****Information and assumptions**

Provided in problem statement

**Find**

How much power must pump 2 add to the water.

**Solution**

Energy equation between (1) and (2):

$$\frac{p_1}{\gamma} + z_1 + \frac{V_1^2}{2g} + h_p - h_L = \frac{p_2}{\gamma} + z_2 + \frac{V_2^2}{2g} \quad (+3)$$

Where

$$p_1 = 0, \quad p_2 = 0, \quad V_1 = 0, \quad z_1 = 0, \quad z_2 = 2m \quad (+2)$$

Thus

$$h_p = h_L + z_2 + \frac{V_2^2}{2g} \quad (+1)$$

The flow rate:  $Q = A_2 V_2 = 0.01 \times 6 = 0.06 \text{ m}^3/\text{s}$

The head loss:  $h_L = 250Q^2 = 250 \times 0.06^2 = 0.9m$

Thus

$$h_p = 0.9 + 2 + \frac{6^2}{2 \times 9.81} = 4.73m \quad (+2)$$

So that

$$\dot{W}_p = \gamma Q h_p = (9800) \times 0.06 \times 4.73 = 2780W = 2.78kW \quad (+1)$$

Therefore

$$\dot{W}_p = \dot{W}_{pump1} + \dot{W}_{pump2} = 1 + \dot{W}_{pump2} = 2.78$$

$$\dot{W}_{pump2} = 1.78kW \quad (+1)$$

## Prob. 3

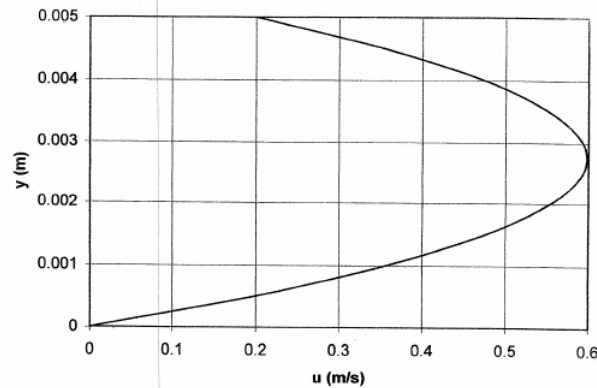
## Information and assumptions

Provided in problem statement

## Find

- (a) Calculate the velocity at  $y=2.5\text{mm}$ .  
 (b) Calculate the shear stress on the bottom plate.

## Solution



- (a) The velocity distribution:

$$\begin{aligned}
 u &= U \frac{y}{b} + \frac{1}{2\mu} \left( \frac{\partial p}{\partial x} \right) (y^2 - by) && (+3) \\
 &= 0.2 \frac{0.0025}{0.005} + \frac{1}{2 \times 0.38} (-60000) (0.0025^2 - 0.005 \times 0.0025) \\
 &= 0.593 \text{ m/s} && (+2)
 \end{aligned}$$

- (b) The shear stress on the bottom plate:

$$\begin{aligned}
 \tau &= \mu \left. \frac{du}{dy} \right|_{y=0} = \mu \left[ \frac{U}{b} + \frac{1}{2\mu} \left( \frac{\partial p}{\partial x} \right) (2y - b) \right] \Big|_{y=0} && (+4) \\
 &= \left[ \mu \frac{U}{b} + \frac{1}{2} \left( \frac{\partial p}{\partial x} \right) (2y - b) \right]_{y=0} \\
 &= \mu \frac{U}{b} + \frac{1}{2} \left( -\frac{\partial p}{\partial x} \right) b \\
 &= 0.38 \times \frac{0.2}{0.005} + \frac{1}{2} (60000) \times 0.005 \\
 &= 165.2 \text{ N/m}^2 && (+1)
 \end{aligned}$$

**Prob. 4****Information and assumptions**

Provided in problem statement

**Find**

At what depth and flowrate would the model operate?

**Solution**

For Froude number similarity:

$$\frac{V_m}{\sqrt{gl_m}} = \frac{V_p}{\sqrt{gl_p}}$$

$$\frac{V_m}{V_p} = \sqrt{\frac{l_m}{l_p}}$$

(+5)

Since the flow rate is  $Q = VA$ , where A is the appropriate cross sectional area,

$$\frac{1}{250} = \frac{Q_m}{Q_p} = \frac{V_m A_m}{V_p A_p} = \sqrt{\frac{l_m}{l_p}} \frac{A_m}{A_p} = \sqrt{\frac{l_m}{l_p}} \left(\frac{l_m}{l_p}\right)^2 = \left(\frac{l_m}{l_p}\right)^{5/2}$$

$$\frac{l_m}{l_p} = 0.11$$

(+3)

So the corresponding model depth

$$l_m = 0.11l_p = 0.11 \times 4 = 0.44 \text{ ft}$$

(+1)

So the corresponding model flowrate

$$Q_m = \frac{1}{250} Q_p = \frac{1}{250} \times 700 = 2.8 \text{ ft}^3/\text{s}$$

(+1)