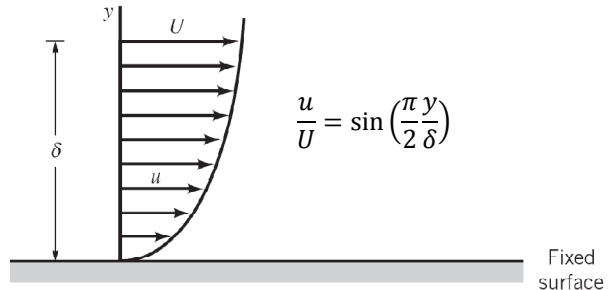


## EXAM1 Solutions

### Problem 1: Shear stress (Chapter 1)

#### Information and assumptions

- Newtonian fluid
- $SG = 0.92$ ,  $\rho_{H_2O@4^\circ C} = 1000 \text{ kg/m}^3$
- $\nu = 4 \times 10^{-4} \text{ m}^2/\text{s}$
- $U = 1 \text{ m/s}$ ,  $\delta = 2 \text{ mm} = 0.002 \text{ m}$
- $\frac{u}{U} = \sin\left(\frac{\pi y}{2\delta}\right)$



#### Find

- The magnitude and direction of the shear stress developed on the plate

#### Solution

(a) Shear stress

$$\tau = \mu \left. \frac{du}{dy} \right|_{y=0} \quad (+6 \text{ points})$$

$$= \mu \frac{d}{dy} \left( U \sin\left(\frac{\pi y}{2\delta}\right) \right) \Big|_{y=0}$$

$$= \mu \frac{\pi U}{2\delta} \cos\left(\frac{\pi y}{2\delta}\right) \Big|_{y=0} = \frac{\pi U}{2\delta} \mu = \frac{\pi U}{2\delta} (SG \cdot \rho_{H_2O} \cdot \nu) \quad (+3 \text{ points})$$

$$= \frac{\pi}{2} \times \frac{1 \text{ m/s}}{0.002 \text{ m}} \times (0.92 \times 1000 \text{ kg/m}^3 \times 4 \times 10^{-4} \text{ m}^2/\text{s})$$

$$= \mathbf{289.0 \text{ N/m}^2} \quad (+0.5 \text{ point})$$

(b) Direction: acting to **right** on plate

(+0.5 point)

## EXAM1 Solutions

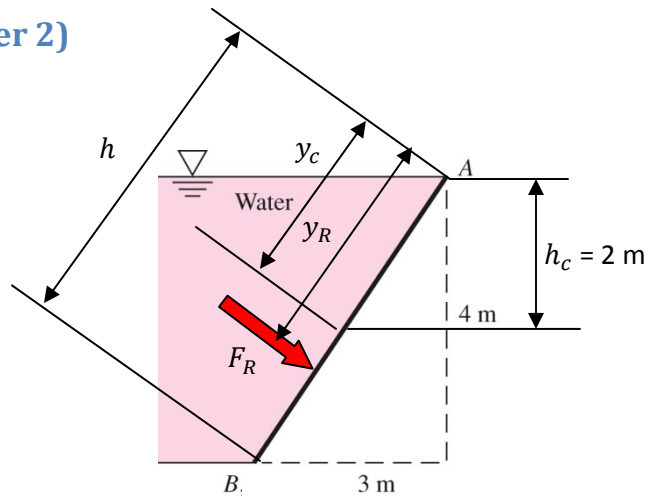
### Problem 2: Hydrostatic force (Chapter 2)

#### Information and assumptions

- $\gamma_{\text{water}} = 9790 \text{ N/m}^3$
- panel width  $b = 2 \text{ m}$
- Slant angle:  $\tan \theta = 4/3$
- $I_{xx} = \frac{bh^3}{12}$

#### Find

- Water force acting on the panel
- Line of action (center of pressure)



#### Solution

(a) Water force  $F_R$

$$F_R = \bar{p}A = \gamma h_c A = \gamma h_c (bh) \quad (+3.5 \text{ points})$$

$$= 9790 \text{ N/m}^3 \times 2 \text{ m} \times (2 \text{ m} \times \sqrt{3^2 + 4^2} \text{ m})$$

$$= 195,800 \text{ N} = \mathbf{196 \text{ kN}} \quad (+0.5 \text{ point})$$

(b) Center of pressure

$$y_R = \frac{I_{xc}}{y_c A} + y_c = \frac{bh^3/12}{(h/2)(bh)} + \frac{h}{2} \quad (+5.5 \text{ points})$$

$$= \frac{2 \text{ m} \times (5 \text{ m})^3 / 12}{(5/2 \text{ m}) \times 10 \text{ m}^2} + \frac{5}{2} \text{ m}$$

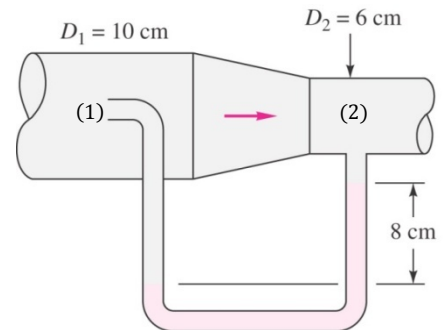
$$= \mathbf{3.33 \text{ m}} \quad (+0.5 \text{ point})$$

**EXAM1 Solutions****Problem 3: Bernoulli equation and manometer (Chapter 3)****Information and assumptions**

- $\rho_{H_2O} = 998.2 \text{ kg/m}^3$  at  $20^\circ\text{C}$
- $\rho_M = 1360 \text{ kg/m}^3$  at  $20^\circ\text{C}$
- $p_1 = 170 \text{ kPa}$
- Losses in the manometer are negligible
- Steady, inviscid, incompressible flow

**Find**

- $p_2$  and flow rate  $Q$

**Solution**

Manometer

$$p_2 = p_1 - (\rho_M - \rho_{H_2O})gh \quad (+1.5 \text{ points})$$

$$= 170,000 \text{ Pa} - (1360 - 998.2) \text{ Kg/m}^3 \times 9.81 \text{ m/s}^2 \times 0.08 \text{ m}$$

$$= \mathbf{169,716 \text{ Pa}} \quad (+0.5 \text{ points})$$

Bernoulli equation

$$p_1 + \frac{\rho}{2}V_1^2 + \rho gz_1 = p_2 + \frac{\rho}{2}V_2^2 + \rho gz_2$$

Since  $V_1 = 0$  and  $z_1 = z_2$ ,

$$V_2 = \sqrt{\frac{2(p_1 - p_2)}{\rho_{H_2O}}} \quad (+6 \text{ points})$$

Flow rate

$$Q = V_2 A_2 = \sqrt{\frac{2(p_1 - p_2)}{\rho_{H_2O}}} \times \frac{\pi}{4} D_2^2 \quad (+1.5 \text{ points})$$

$$= \sqrt{\frac{2 \times (170,000 - 169,716) \text{ Pa}}{998.2 \text{ Kg/m}^3}} \times \frac{\pi}{4} \times (0.06 \text{ m})^2$$

$$= 0.0021 \text{ m}^3/\text{s}$$

$$= \mathbf{7.6 \text{ m}^3/\text{hr}} \quad (+0.5 \text{ point})$$

## EXAM1 Solutions

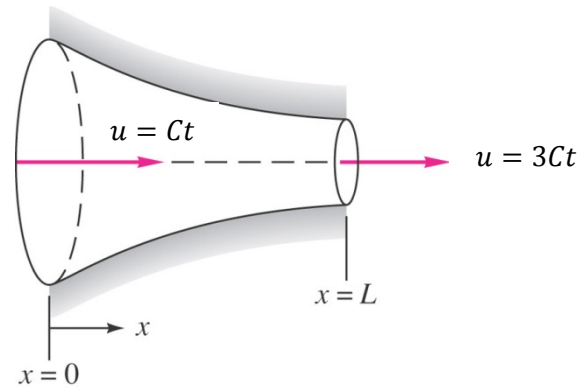
## Problem 4: Acceleration and Euler equation (Chapter 4)

## Information and assumptions

- One-dimensional flow
- $u = C \left(1 + \frac{2x}{L}\right) t$ ,  $C = 1 \text{ ft/s}^2$ ,  $L = 6 \text{ in}$
- Viscosity effects are negligible
- $\rho = 1.0 \text{ lbm/ft}^3$

## Find

- Local and convective accelerations and pressure gradient at  $x = \frac{L}{2}$  and  $t = 1 \text{ s}$



## Solution

(a) Local acceleration

$$\frac{\partial u}{\partial t} = \frac{\partial}{\partial t} \left[ C \left(1 + \frac{2x}{L}\right) t \right] = C \left(1 + \frac{2x}{L}\right) \quad (+3.5 \text{ points})$$

$$\left. \frac{\partial u}{\partial t} \right|_{x=L/2, t=1} = 1 \frac{\text{ft}}{\text{s}^2} \times \left(1 + \frac{2 \times L/2}{L}\right) = \mathbf{2 \text{ ft/s}^2} \quad (+0.5 \text{ point})$$

(b) Convective acceleration

$$u \frac{\partial u}{\partial x} = \left[ C \left(1 + \frac{2x}{L}\right) t \right] \times \frac{\partial}{\partial x} \left[ C \left(1 + \frac{2x}{L}\right) t \right] = \frac{2C^2 t^2}{L} \left(1 + \frac{2x}{L}\right) \quad (+3.5 \text{ points})$$

$$\left. u \frac{\partial u}{\partial x} \right|_{x=L/2, t=1} = \frac{2 \times (1 \text{ ft/s}^2)^2 \times (1 \text{ s})^2}{1/2 \text{ ft}} \times \left(1 + \frac{2 \times L/2}{L}\right) = \mathbf{8 \text{ ft/s}^2} \quad (+0.5 \text{ points})$$

(c) Pressure gradient

$$\begin{aligned} \frac{dp}{dx} &= -\rho \left( \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} \right) \\ &= -\rho \left[ C \left(1 + \frac{2x}{L}\right) + \frac{2C^2 t^2}{L} \left(1 + \frac{2x}{L}\right) \right] \\ &= -\rho C \left(1 + \frac{2x}{L}\right) \left(1 + \frac{2Ct^2}{L}\right) \quad (+1.5 \text{ points}) \end{aligned}$$

$$\begin{aligned} \left. \frac{dp}{dx} \right|_{x=L/2, t=1} &= -(1 \text{ lbm/ft}^3 \times 1 \text{ ft/s}^2) \times \left(1 + \frac{2 \times L/2}{L}\right) \times \left(1 + \frac{2 \times 1 \text{ ft/s}^2 \times (1 \text{ s})^2}{1/2 \text{ ft}}\right) \\ &= \mathbf{-10 \text{ lbm/ft}^2 \cdot \text{s}^2} \quad (+0.5 \text{ point}) \end{aligned}$$