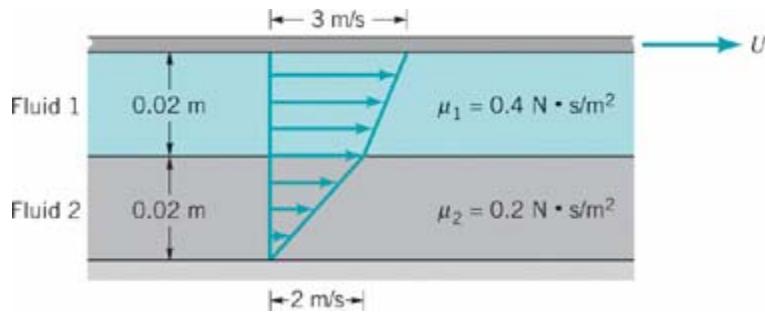


**Prob. 1****Information and assumptions****Find**

Determine the ratio of these two shear stresses.

**Solution**

For fluid 1:

$$\tau_1 = \mu_1 \left( \frac{du}{dy} \right)_{top \text{ surface}} = \left( 0.4 \frac{N \cdot s}{m^2} \right) \left( \frac{3 - 2}{0.02} \frac{m/s}{m} \right) = 20 \frac{N}{m^2} \quad (+4.5)$$

For fluid 2:

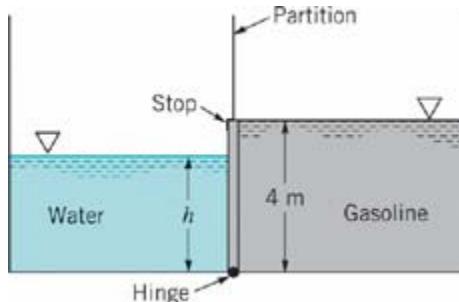
$$\tau_2 = \mu_2 \left( \frac{du}{dy} \right)_{bottom \text{ surface}} = \left( 0.2 \frac{N \cdot s}{m^2} \right) \left( \frac{2 - 0}{0.02} \frac{m/s}{m} \right) = 20 \frac{N}{m^2} \quad (+4.5)$$

Thus:

$$\frac{\tau_2}{\tau_1} = \frac{20 \frac{N}{m^2}}{20 \frac{N}{m^2}} = 1 \quad (+1)$$

**Prob. 2****Information and assumptions**

Provided in problem statement

**Find**Determine water depth  $h$ .**Solution**

For water:

$$h_{cw} = \frac{h}{2}$$

$$F_{Rw} = \gamma_w h_{cw} A_w = \gamma_w \left(\frac{h}{2}\right) (2h) = \gamma_w h^2$$

$$y_{Rw} = \frac{I_{xc}}{y_c A} + y_c = \frac{2h^3}{12(h/2)(2h)} + \frac{h}{2} = \frac{2h}{3} \quad (+4)$$

For gasoline:

$$h_{cg} = 2$$

$$F_{Rg} = \gamma_g h_{cg} A_g = \gamma_g (2)(2 \times 4) = 16\gamma_g$$

$$y_{Rg} = \frac{I_{xc}}{y_c A} + y_c = \frac{2 \times 4^3}{12(2)(2 \times 4)} + 2 = \frac{8}{3}m \quad (+4)$$

Taking the moment about the hinge:

$$F_{Rw}(h - y_{Rw}) = F_{Rg}(4 - y_{Rg})$$

$$\gamma_w h^2 \left(h - \frac{2}{3}h\right) = 16\gamma_g \left(4 - \frac{8}{3}\right)$$

$$\gamma_w \frac{1}{3}h^3 = \frac{64}{3}\gamma_g$$

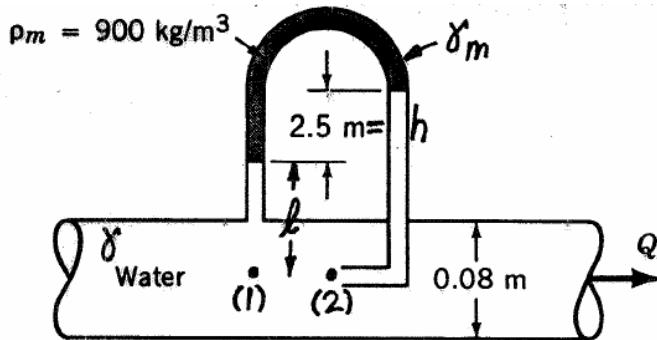
$$h = \left(\frac{64\gamma_g}{\gamma_w}\right)^{\frac{1}{3}} = \left(\frac{64 \times 700}{1000}\right)^{\frac{1}{3}} = 3.55m \quad (+2)$$

**Prob. 3****Information and assumptions**

Provided in problem statement

**Find**

Determine the flowrate through the pipe.

**Solution**

The Bernoulli equation between points (1) and (2):

$$\frac{p_1}{\gamma} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\gamma} + \frac{V_2^2}{2g} + z_2 \quad (+5)$$

Where  $z_1 = z_2$  and  $V_2 = 0$ 

Thus

$$\begin{aligned} \frac{p_1}{\gamma} + \frac{V_1^2}{2g} &= \frac{p_2}{\gamma} \\ V_1 &= \sqrt{2g \frac{p_2 - p_1}{\gamma}} \end{aligned} \quad (+1)$$

The manometer equation:

$$\begin{aligned} p_1 - \gamma l - \gamma_m h + \gamma(l + h) &= p_2 \\ p_2 - p_1 &= (\gamma - \gamma_m)h \end{aligned} \quad (+2)$$

So that

$$V_1 = \sqrt{2g \left(1 - \frac{\gamma_m}{\gamma}\right)h} = \sqrt{2 \times 9.81 \times \left(1 - \frac{900}{1000}\right) \times 2.5} = 2.21 \text{ m/s} \quad (+1)$$

Thus

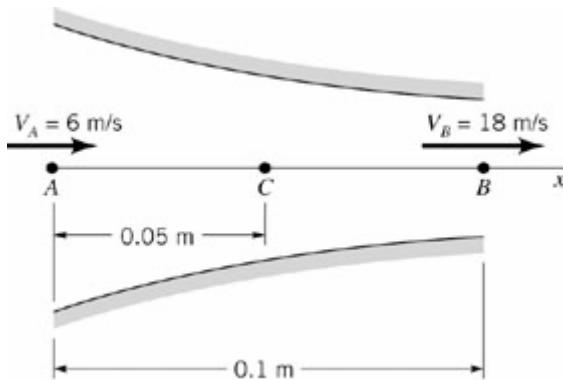
$$Q = A_1 V_1 = \frac{\pi}{4} \times (0.08)^2 \times 2.21 = 0.0111 \text{ m}^3/\text{s} \quad (+1)$$

**Prob. 4****Information and assumptions**

Provided in problem statement

**Find**

Determine the acceleration at points A, B, and C.

**Solution**

The acceleration

$$\mathbf{a} = \frac{D\mathbf{V}}{Dt} = \frac{\partial \mathbf{V}}{\partial t} + \mathbf{V} \cdot \nabla \mathbf{V} \quad (+5)$$

With  $u = u(x)$ ,  $v = 0$ , and  $w = 0$ 

$$\mathbf{a} = \left( \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} \right) \mathbf{i} = u \frac{\partial u}{\partial x} \mathbf{i} \quad (+1)$$

Since  $u$  is a linear function of  $x$ ,  $u = c_1x + c_2$ , where the constants  $c_1, c_2$  are given as:

$$u_A = 6 = c_2 \quad (+1)$$

$$u_B = 18 = 0.1c_1 + c_2 \quad (+1)$$

i.e.,  $c_1 = 120$ ,  $c_2 = 6$ 

$$\text{Thus } u = 120x + 6, \frac{\partial u}{\partial x} = 120 \quad (+1)$$

$$\text{For } x_A = 0.0m, u_A = 6 \text{ m/s}, \mathbf{a}_A = 6 \times 120 \mathbf{i} = 720 \mathbf{i} \text{ m/s}^2 \quad (+1)$$

$$\text{For } x_C = 0.05m, u_C = 12 \text{ m/s}, \mathbf{a}_C = 12 \times 120 \mathbf{i} = 1440 \mathbf{i} \text{ m/s}^2 \quad (+1)$$

$$\text{For } x_B = 0.1m, u_B = 18 \text{ m/s}, \mathbf{a}_B = 18 \times 120 \mathbf{i} = 2160 \mathbf{i} \text{ m/s}^2 \quad (+1)$$