57:020 Mechanics of Fluids and Transfer Processes **Exercise Notes for the Pipe Flow TM**

Measurement of Flow Rate, Velocity Profile and Friction Factor in Pipe Flows S. Ghosh, M. Muste, M. Wilson and F. Stern

1. Purpose

To provide students *hands-on* experience with Pipe stand test facility and modern measurement systems including pressure transducers and pitot probes and computerized data acquisition using Labview. To measure flow rate, velocity profiles and friction factors in smooth and rough pipes, determine the measurement uncertainties, and compare the results with benchmark data.

2. Experimental Design

The experiments are conducted in an instructional airflow pipe facility (Figure 1). The air is blown into a large reservoir located at the upstream end of the system. Pressure built up in the reservoir, forces the air to flow through any of the three horizontal pipes. Pressure taps are located on each pipe along intervals of 1.524m, for static pressure measurements. The pipe characteristics for each of the pipes are provided in Appendix A. At the downstream end of the system, the air is directed downward and back, through any of the three pipes of varying diameters fitted with Venturi meters (Figure 2). The top three valves control flow through the experimental pipes, while the bottom three valves control the Venturi meter to be used. The venture meter with 5.08cm diameter is used to measure the total flow rate, the other two are kept closed. Six gate valves are used for directing the flow. The top and bottom 5.08cm pipes are only used for measurements while the middle one is kept closed during the experiment. Velocity measurements in the top and bottom pipes are obtained using pitot probe (Figure 3).

Figure1. Airflow pipe system Figure 2. Venturimeter Figure 3. Pitot-probe

Pressures are acquired either manually, using simple and differential manometers for data acquisition, or automatically whereby the manometers are connected to an automated Data Acquisition (DA) system that converts pressure to voltages using pressure transducers. Data acquisition is controlled and interfaced by Labview software described in Appendix B. The schematic of the two alternative measurement systems is provided in Figure 4.

All pressure taps on the pipes, venturimeters and pitot probes have 0.635cm diameter quick coupler connections that can be hooked up to the pressure transducers.

2.1 Data reduction (DR) equations

In a fully developed, axi-symmetric pipe flow, the axial velocity $(u = u (r))$ at a radial distance *r* from the pipe centerline, is independent of the direction in which *r* is considered (Figure 5). However, the shape of the velocity profile is different for laminar and turbulent flows.

Laminar and turbulent flow regimes are distinguished by the flow Reynolds number defined as

$$
Re = \frac{VD}{v} = \frac{4Q}{\pi D v}
$$
 (1)

Where, *V* is the average pipe velocity, *D* is the pipe diameter, *Q* is the pipe flow rate, and *ν* is the kinematic viscosity of the fluid. For fully developed laminar flow (*Re* < 2000), analytical solution for the differential equations of the fluid flow (Navier-Stokes and continuity) can be obtained. For turbulent pipe flows (*Re* > 2000), there is no exact solution, hence semi-empirical laws for velocity distribution are used instead.

 The pipe-head loss due to friction is obtained from the Darcy-Weisbach equation:

Figure 5. Velocity distributions for fully developed pipe flow: a) laminar flow; b) turbulent flow

$$
h_f = f \frac{L V^2}{D 2g} \tag{2}
$$

where, f is the (Darcy) friction factor, L is the length of the pipe over which the loss occurs, h_f is the head loss due to viscous effects, and *g* is the gravitational acceleration. Moody diagram provides the friction factor for pipe flows with smooth and rough walls in laminar and turbulent regimes. The friction factor depends on *Re* and relative roughness *k*/*D* of the pipe (for large enough *Re*, the friction factor is solely dependent on the relative roughness).

Velocity distributions in the pipes are measured with Pitot tubes housed in glass-walled boxes (Figure 3). The data reduction equation (DRE) for the measurement of the velocity profiles is obtained by applying Bernoulli's equation for the Pitot tube

$$
u(r) = \left[\frac{2 \cdot g \rho_w}{\rho_a} \cdot \left[z_{SM_{Sug}}(r) - z_{SM_{Sud}}\right]\right]^{1/2}
$$
(3)

where, $u(r)$ is the velocity at the radial position *r*, *g* is the gravitational acceleration, $z_{SM_{Sue}}(r)$ is the stagnation pressure head determined by the Pitot probe located at radial position r , Z_{SM} is the static pressure head in the pipe, equal to that of the ambient pressure inside the glass-walled box. The readings of the pressure heads in Equation (3) are in height of a liquid column (ft of water). ρ_w , is the density of water and ρ_a is the density of air. DRE for the friction factor is one of the Darcy Weisbach equation forms (Roberson & Crowe, 1997)

$$
f = \frac{g\pi^2 D^5}{8LQ^2} \frac{\rho_w}{\rho_a} \left(z_{SM\,i} - z_{SM\,j} \right) \tag{4}
$$

where, *L* is the pipe length between the taps *i* and *j*, and $z_{SM} - z_{SM}$ *j* is the difference in pressure between the taps *i* and *j*. The flow rate *Q* is directly measured using the calibration equations for the Venturi meters (Rouse, 1978)

$$
Q = C_d A_t \sqrt{2g\Delta z_{DM} \cdot \frac{\rho_w}{\rho_a}}
$$
 (5)

where, C_d is the discharge coefficient, A_t is the contraction area, Δz_{DM} is the head drop across the Venturi, measured in height of liquid column (ft of water) by the differential manometer or the pressure transducer. Appendix A lists Venturi meter characteristics. Alternatively, the flow rate can be determined by integrating the measured velocity distribution over the pipe cross-section.

$$
Q_i = 2\pi \int_0^r u(r)r dr \tag{6}
$$

3. Experimental Process

Figure6. EFD Process

3.1. Test-setup

The experimental measurement systems for the manual and automated configurations are shown below:

3.2. Data Acquisition

Each student group will obtain velocity distributions and determine the friction factor for one of the 5.08cm (rough or smooth) pipes. Data acquired with the DA are recorded electronically and subsequently used for data reduction. The **Data reduction sheet** will be used for data reduction. The experimental procedure follows the sequence described below:

1. Starting with the low velocity initially set, increase gradually the flow rate until the desired *Re (= 96,000)* in the test section is attained (the desired *Re* can be achieved for both upper and lower pipes, with a setting of 35% on the blower motor controller and control valves fully open). Make at least five flow rate measurements with the venture-

meter of 5.08cm contraction diameter. The other two venture-meters are kept closed. For the five measurements, the blower setting should start from 15% and reach 35%, with 5% increment each time (recommended settings: 15%, 20% 25% 30% 35%). Record your readings from both the manometer and pressure transducer. The remaining experiment is carried out at 35% blower settings. Use Labview to record the venturimeter reading.

- 2. Take temperature readings with the digital thermometer (resolution 0.1 ˚F) for ambient air and inside the pipe for calculating the corresponding water and air densities, respectively. Input the temperature readings as requested by DA software interface. Since the temperature increases during the experiment, take three temperature readings at the beginning, in the middle, and at the end of the measurements.
- 3. Velocity distribution is obtained with the DA by measuring stagnation heads across the full pipe diameter along with the readings of the static heads using the appropriate Pitot-tube assembly. Measure stagnation heads at radial intervals no greater than 5 mm (recommended spacing for half diameter of the upper and lower pipes is 0, 5, 10, 15, 20, 23, and 24 mm). Positioning of the Pitot tube within the pipe is made with a micrometer (resolution of 0.01 mm). To establish precision limits for velocity profiles, measurements near the pipe wall (at 24mm) should be taken at least 10 times. The same procedure is used for the upper (smooth) and lower (rough) pipes.
- 4. Keeping the blower setting at 35%, measure the pressure heads at pressure taps 1, 2, 3, and 4 sequentially as indicated in Figure 1 using the DA by connecting each tap to the pressure transducer. To establish precision limits for the friction factor, measurements, preferably at taps 3 and 4 should be repeated 10 times. The repeated measurements should be made alternatively between tap 3 and 4. It is important to note that the pressure in the pipe system fluctuates when opening or closing manifold valves, hence it is necessary to wait a few seconds between consecutive measurements, for the pressure fluctuation to settle down. The data acquisition procedure is same for both rough and smooth pipes.

3.3 Data Reduction

Data reduction includes the following steps:

- **1.** Using the average temperatures, \overline{T}_w and \overline{T}_a to determine ρ_w , ρ_a , and v_a from fluid property tables. Determine the flow rate (*Q*) in the individual pipes using Equation (6) and *Re* using Equation (1)*.* The method for calculating flow rate (Equation 6) in individual pipes is explained in the **Data reduction sheet.**
- 2. Compare the flow rate readings taken with the manometer and pressure transducer.
- 3. Calculate velocity distribution profiles for the tested pipe using Equation (3). Plot the measured velocity profile including the velocity total uncertainties calculated for centerline and near the wall measurements. Compare the measured velocity distribution with the benchmark data provided in Figure 8.
- 4. Calculate the friction factor for the tested pipe using Equation (4). Use readings at taps 3 and 4, where the flow is fully developed. Compare *f* with benchmark data, including uncertainty band for the measured *f*.

3.4 Uncertainty Analysis

Uncertainties for the experimentally measured velocities and friction factor will be evaluated. The methodology for estimating uncertainties follows the AIAA S-071 Standard (AIAA, 1995) as summarized in Stern et al. (1999) for multiple tests $(M = 10)$. The block diagrams for error propagations in the measurements are provided in Figure 7. Elemental errors for each of the measured independent variable in data reduction equations should be identified using the best available information (for bias errors) and repeated measurements (for precision errors). We will consider in the analysis only the largest bias limits and neglect correlated bias errors. The spreadsheet for evaluating the uncertainties is provided in **Data reduction sheet**. The spreadsheet includes bias limit estimates for the individual measured variables.

The DRE for the velocity profile, Equation (3), is of the form: $u(r) = F(g, \rho_w, \rho_a, z_{SM, stag}, z_{SM, stat})$. We will

only consider bias limits for *zSM stag* and *zSM stat*. The total uncertainty for velocity measurements is

$$
U_u^2 = B_u^2 + P_u^2 \tag{7}
$$

The bias limit, B_u , and the precision limit, P_u , for velocities are given by

$$
B_u^2 = \sum_{i=1}^j \theta_i^2 B_i^2 = \theta_{Z_{SM\,stag}}^2 B_{Z_{SM\,stag}}^2 + \theta_{Z_{SM\,stat}}^2 B_{Z_{SM\,stat}}^2
$$
 (8)

$$
P_u = KS_u / \sqrt{M} \tag{9}
$$

where the coefficients θ are calculated using mean values for the independent variables

$$
\Theta_{Z_{SM\,\text{stag}}} = \frac{1}{(z_{SM\,\text{stag}} - z_{SM\,\text{stat}})^{0.5}} \left(0.5 g \frac{\rho_w}{\rho_a} \right)^{0.5} \left(s^{-1} \right), \quad \Theta_{Z_{SM\,\text{stat}}} = \frac{-1}{(z_{SM\,\text{stag}} - z_{SM\,\text{stat}})^{0.5}} \left(0.5 g \frac{\rho_w}{\rho_a} \right)^{0.5} \left(s^{-1} \right) (10)
$$

and S_u is the standard deviation of the repeated velocity measurements. $K = 2$ for $(M =) 10$ repeated measurements.

The DRE for the friction factor, Equation (4), is of the form: $f = F(g, D, L, Q, \rho_w, \rho_a, z_{SM,i}, z_{SM,j})$. We will only consider bias limits for z_{SM} and z_{SM} . The total uncertainty for the friction factor is:

$$
U_f^2 = B_f^2 + P_f^2 \tag{11}
$$

The bias limit, B_f , and the precision limit, P_f , for the result are given by

$$
B_f^2 = \sum_{i=1}^j \theta_i^2 B_i^2 = \theta_{z_{SMi}}^2 B_{z_{SMi}}^2 + \theta_{z_{SMj}}^2 B_{z_{SMj}}^2
$$
 (12)

$$
P_f = KS_f / \sqrt{M} \tag{13}
$$

where, the coefficients θ are calculated using mean values for the independent variables:

$$
\theta_{Z_{SMi}} = \frac{g\pi^2 D^5}{8LQ^2} \frac{\rho_w}{\rho_a} (1) \quad (m^{-1}) \qquad \theta_{Z_{SMj}} = \frac{g\pi^2 D^5}{8LQ^2} \frac{\rho_w}{\rho_a} (-1) \quad (m^{-1}) \qquad (14)
$$

and S_f is the standard deviation of the repeated friction factor measurements. $K = 2$ for ($M =$) 10 repeated measurements.

a)

Figure 7. Block diagrams for uncertainty estimation: a) velocity; b) friction factor

| Variable | Bias Limit | |
|---|------------------------|-----------|
| | Estimation based on: | Value |
| $D_{z_{SM}}^{\qquad}, B_{z_{SM}}^{\qquad}$ (m) | Last significant digit | 0.0003048 |
| $B_{Z_{SM\;staguation}}, B_{Z_{SM\;state}}(\textrm{m})$ | Last significant digit | 0.0003048 |

Table1. Bias limits for the individual variables included in the data reduction equations

3.5. Data Analysis

Measurements obtained in the experiments will be compared with benchmark data. The benchmark data for velocity distribution is provided in numerical and graphical form in Figure 8. The benchmark data for friction factor is provided by the Moody diagram (Figure 9) and by the Colebrook-White-based formula (Roberson and Crowe, 1997)

$$
f = \frac{0.25}{\left[\log \left(\frac{k/D}{3.7} + \frac{5.74}{\text{Re}^{0.9}} \right) \right]^2}
$$
(15)

The following questions help to evaluate fluid physics, EFD process, and uncertainty analysis. The solutions to these questions **must be included** in the Data analysis section of the lab report. Use **data reduction sheet** and attach it to your lab reports.

- 1. Comment on the differences in flow rate readings obtained by the manometer and pressure transducer.
- 2. Plot the velocity profile $u(r)$ obtained from the experiment normalized by maximum velocity in pipe (u/U_{max}) against radial distance *r*, normalized by maximum radius (*r/R*). Plot the Schlichting data given in Figure 8 on the same plot. Compare the two profiles. Choose a point near the wall where, the value of *r/R*, is close to 1. Show the total percentage uncertainty at that point using an uncertainty band.
- 3. Plot the head (in ft of air) at each pressure tap as a function of distance along the pipe. Comment on the pressure head drop distribution along the pipe and comment on uncertainties and unaccounted error sources.
- 4. Calculate the friction factor and compare results with the Moody diagram. Show the experimental value of the friction factor on the Moody diagram along with the uncertainty band.
- 5. What is the advantage of using non-dimensional forms for variables such as those shown in Figures (8) and (9)?

profile

friction factor

4. References

Roberson, J.A. and Crowe, C.T. (1997). *Engineering Fluid Mechanics, 7*th edition, Houghton Mifflin, Boston, MA. Schlichting, H. (1968). *Boundary-Layer Theory*, McGraw-Hill, New York, NY.

Rouse, H. (1978). Elementary Mechanics of Fluids, Dover Publications, Inc., New Yoirk, NY.

Stern, F., Muste, M., Beninati, L-M, Eichinger, B. (1999). "Summary of Experimental Uncertainty Assessment Methodology with Example," IIHR Report No. 406, Iowa Institute of Hydraulic Research, The University of Iowa, Iowa City, IA.

APPENDIX A

SPECIFICATIONS FOR THE EXPERIMENTAL FACILITY COMPONENTS

Table A1. Pipe characteristics

a) Photograph of experimental setup b) Schematic of experimental setup

Figure A.1. Layout of the data acquisition systems

APPENDIX B

THE AUTOMATED DATA ACQUISITION SYSTEM (ADAS)

