## November 16, 2015

1. Water flows steadily through the nozzle shown in Fig. 1, discharging to atmosphere. Calculate (a) the jet velocity $V_{2}$ at the nozzle end, (b) the pressure $p_{1}$ at the flanged joint, and (c) the horizontal component of the anchoring force $F_{x}$ to keep the nozzle in place. The elevation difference is 12 in . and no loss between the flanged joint and the nozzle end (i.e., between sections 1 and 2 ). Use $\rho=1.94$ slugs $/ \mathrm{ft}^{3}$ and $\gamma=62.4 \mathrm{lb} / \mathrm{ft}^{3}$ for water and $\mathrm{g}=32.2 \mathrm{ft} / \mathrm{s}^{2}$.


Figure 1
2. A capillary tube of inside diameter $d=6 \mathrm{~mm}$ connects tank $A$ and open container $B$ as shown in Fig. 2. The liquid in $A, B$, and capillary $C D$ is water having a specific weight $\gamma=9,780 \mathrm{~N} / \mathrm{m}^{3}$ and a viscosity of $\mu=0.0008 \mathrm{~N} \cdot \mathrm{~s} / \mathrm{m}^{2}$. The pressure $p_{A}=34.5 \mathrm{kPa}$ gage. Neglecting the minor losses at $C$ and $D$, determine the flow rate $Q$ through the capillary tube. Assume laminar flow from $A$ to $B$ and use $h_{f}=32 \mu L V / \gamma d^{2}$ for the friction loss, where $V$ is the water velocity through the capillary tube.


Figure 2

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3. A viscous liquid flows down an inclined plane surface in a steady, fully developed laminar film of thickness $h$ and width $b$ (out of the paper) as shown in Fig. 3. A useful approximation of the flow is

$$
\mu \frac{d^{2} u}{d y^{2}}=-\rho \mathrm{g}_{x}
$$

where, $\mathrm{g}_{x}=\mathrm{g} \cdot \sin \theta$ is the $x$-component of the gravity acceleration. (a) Derive an expression for the velocity distribution $u(y)$ by integrating the given equation then applying the free-shear (i.e., $d u / d y=0$ ) boundary condition at the top and the no-slip boundary condition at the bottom. (b) If the liquid is SAE 30 oil at $15.6^{\circ} \mathrm{C}\left(\rho=912 \mathrm{~kg} / \mathrm{m}^{3}\right.$ and $\left.\mu=0.38 \mathrm{~N} \cdot \mathrm{~s} / \mathrm{m}^{2}\right)$ and $h=1$ $\mathrm{mm}, b=1 \mathrm{~m}$, and $\theta=15^{\circ}$, find the volume flow rate, $Q=\int_{0}^{h} u(y) b d y$.


Figure 3
4. In some speed ranges, vortices are shed from the rear of bluff cylinders placed across a flow. The vortices alternately leave the top and bottom of the cylinder, as shown in Fig. 4. The vortex shedding frequency, $f$, is thought to dependent on fluid density, $\rho$, and viscosity, $\mu$, cylinder diameter, $d$, and free-stream velocity, $V$. (a) Use dimensional analysis to develop a functional relationship for $f$. (b) Vortex shedding occurs in standard air on two cylinders with diameters $d_{m}$ and $d_{p}$, respectively. If the diameter ratio is $d_{p} / d_{m}=2$, determine the velocity ratio, $V_{p} / V_{m}$, for dynamic similarity, and the ratio of vortex shedding frequencies, $f_{p} / f_{m}$. For part (a), use the MLT unit system.


Figure 4

